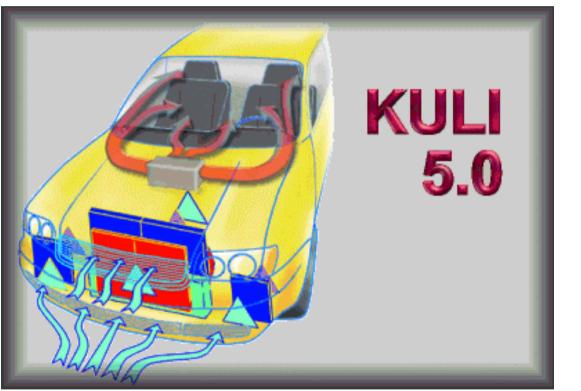
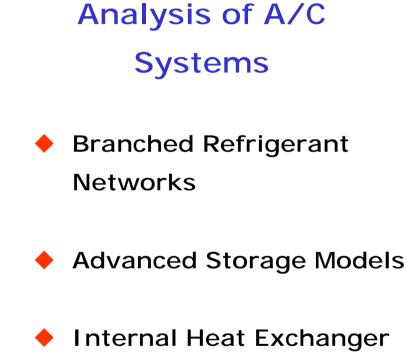
KULI AC







Carbon Dioxide

Thomas Anzenberger, ECS Steyr

Overview

- > Overview KULI AC
- KULI AC, new Components
- Technical Specification
- Practical Application
- Further Development



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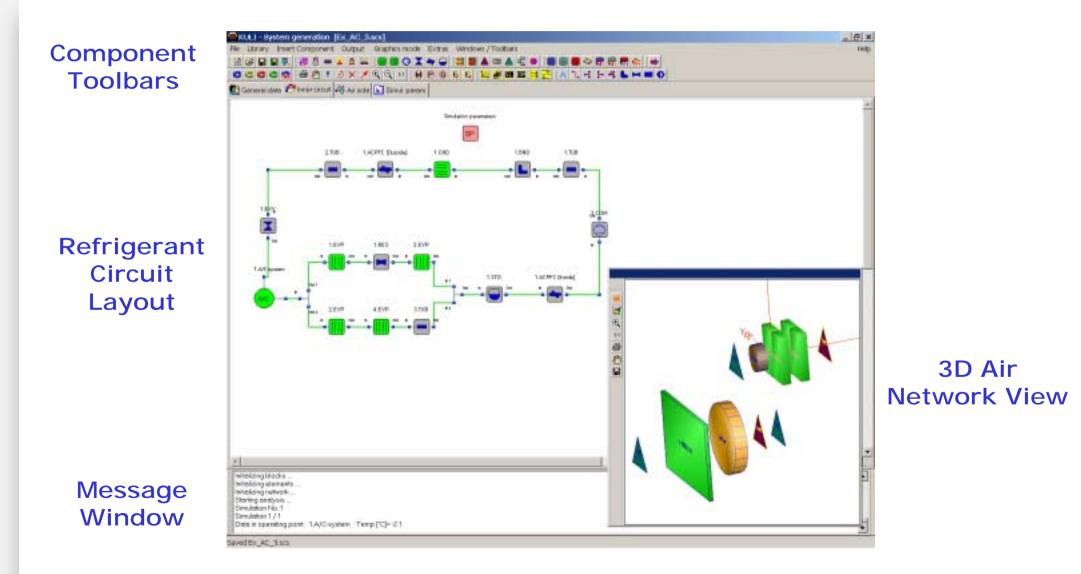
more value more car



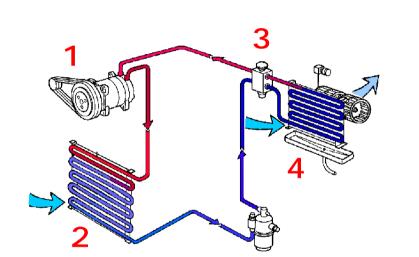
Why A/C Simulation ?



KULI A/C



Refrigerant Circuit

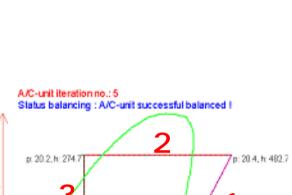


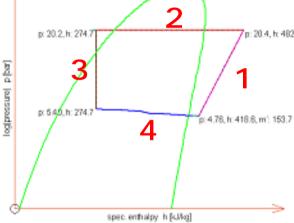
1 Compressor

2 Condenser

3 Expansion Device

4 Evaporator



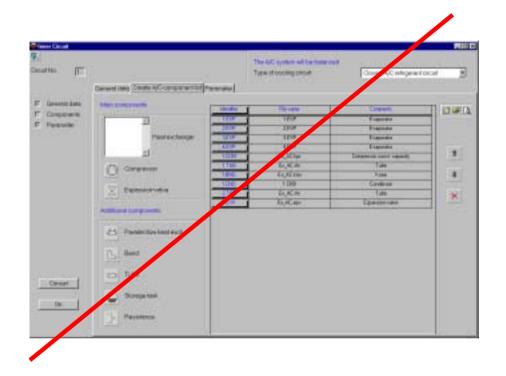


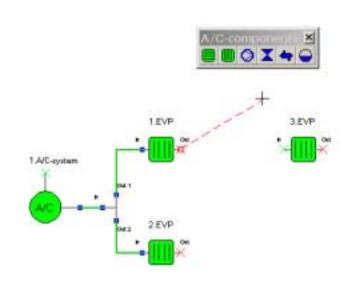
Geometric Model

Analysis Results



Refrigerant Circuit

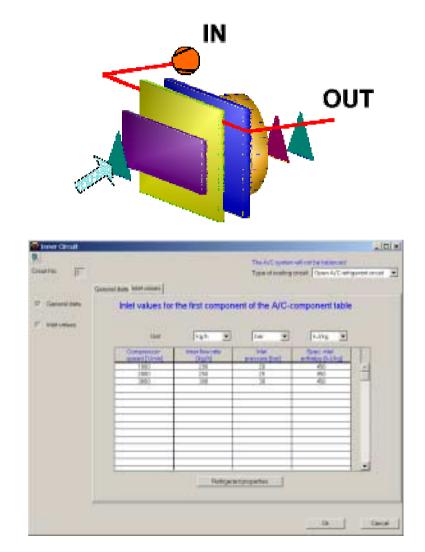




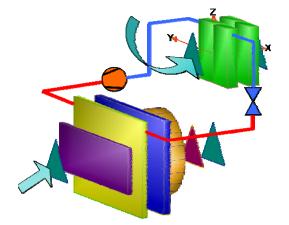
Old: Component Table

New: Component Connection by Clicking

Refrigerant Circuits



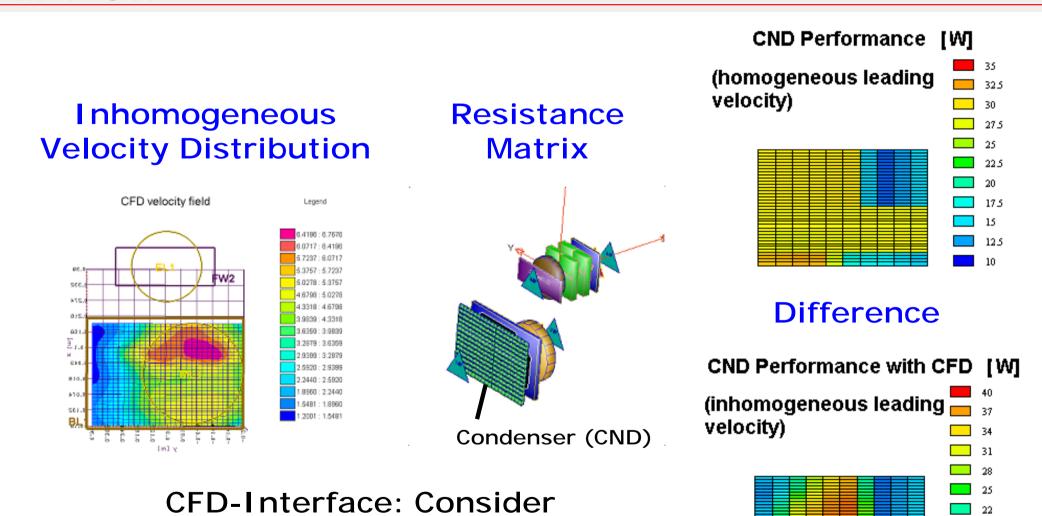
Open: Inlet properties



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urtus IF		The ALC system will be to the end. Type of availing stread (Chosed A/C subspectra stread
	General Buts Parameter	
7 General bah 7 Patemake	C Set a straining	A Secondensistance once door
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	Normal second based variants in reduce the end of the second s	
	His ships many from bendine	
	His always produce and a histophane (4) No always commonlying to contain (7)	

Closed: Equalization

Air Flow



Inhomogeneous Velocity Distributions

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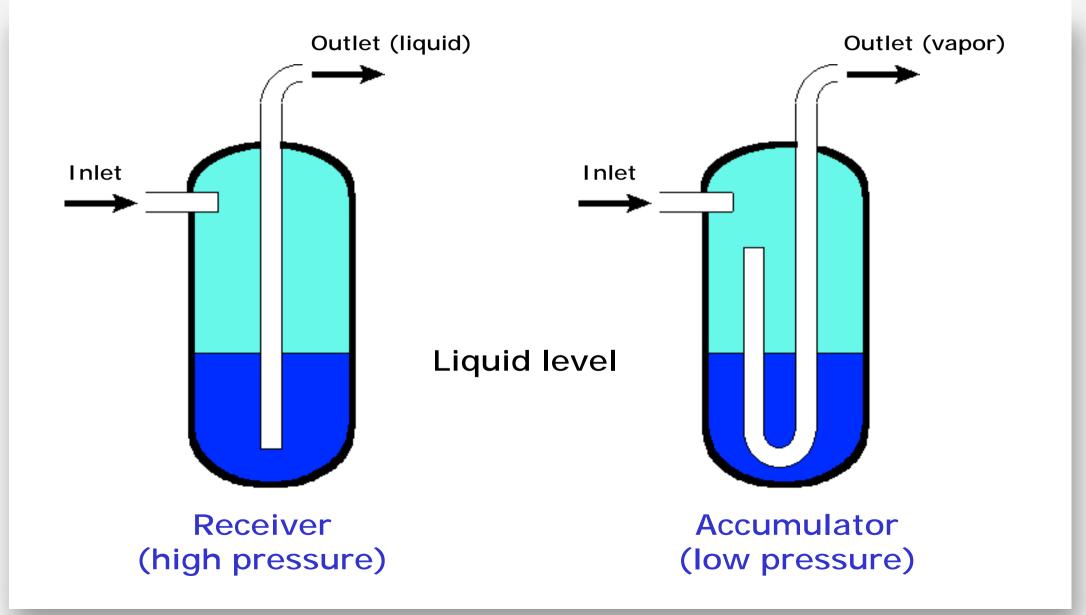
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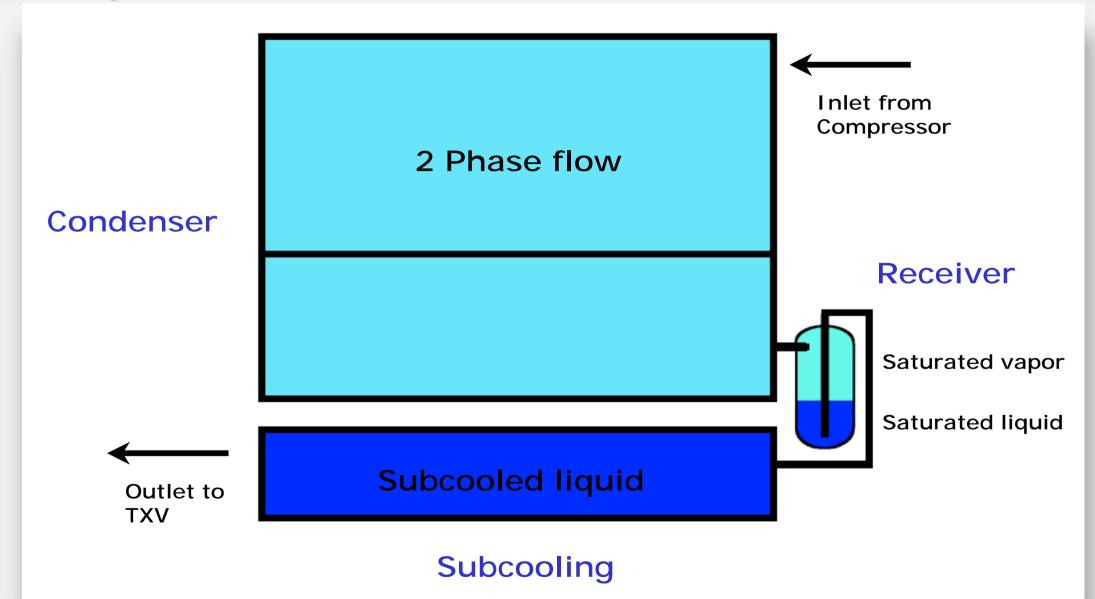
AGNA ST

Storage Model



more value more car

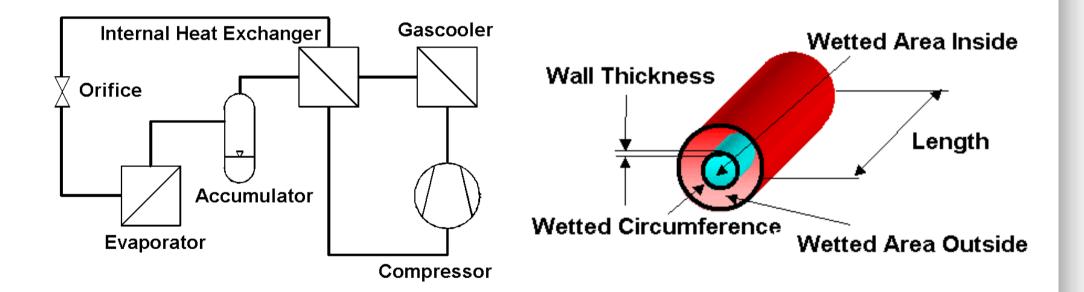
Integrated Receiver





PFC (SLHX)

Internal Heat Exchanger (SLHX)

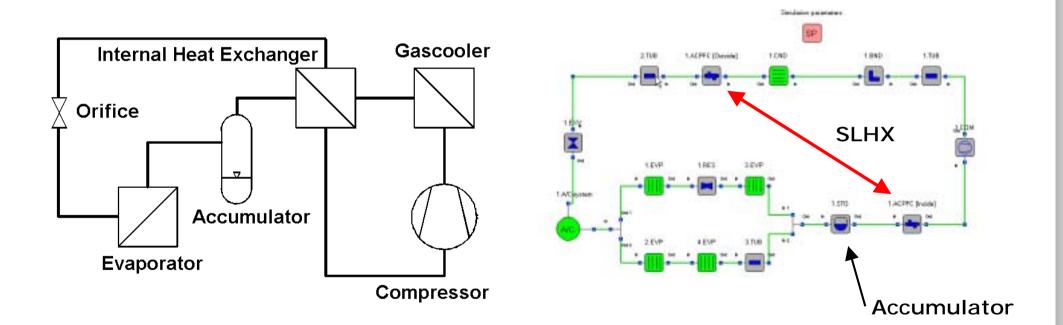


CO₂ Refrigerant Circuit

Geometric Based Model

PFC (SLHX)

Internal Heat Exchanger (SLHX)



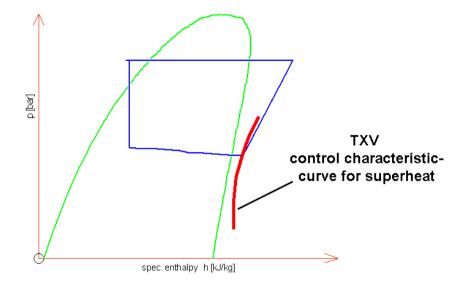
CO₂ Refrigerant Circuit Layout

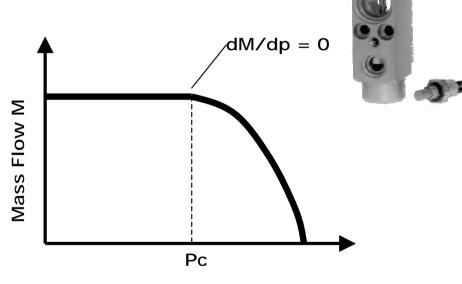
KULI Layout

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Expansion Device

- Thermostatic Controlled Expansion Valve (TXV)
- Uncontrolled Orifice





Outlet pressure Po

TXV: Characteristic Curve for Superheat

Orifice: Critical Mass Flow Calculation

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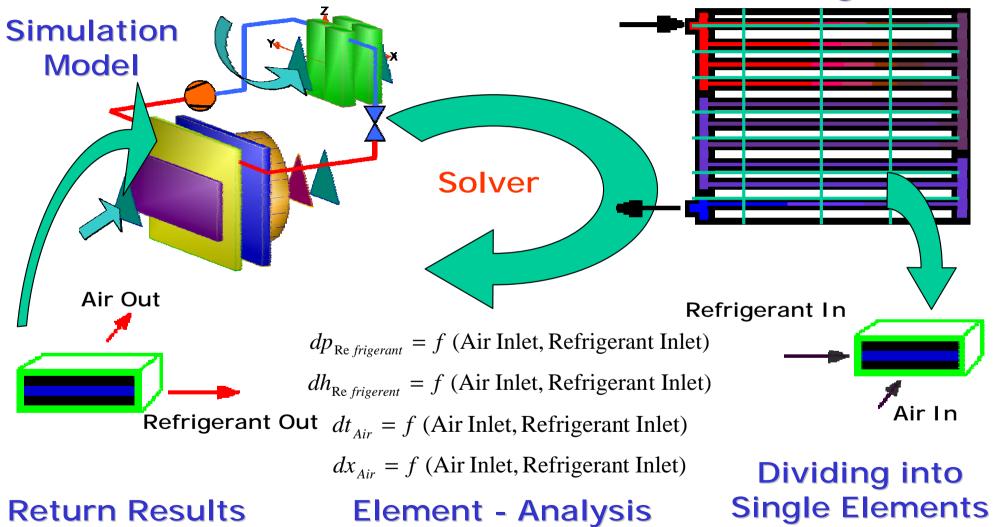


AGNA ST



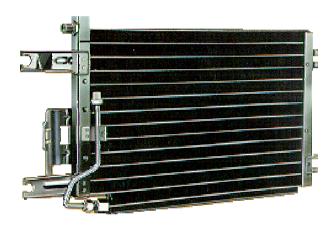


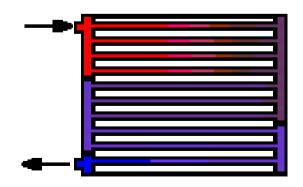




Condenser / Evaporator

Hardware



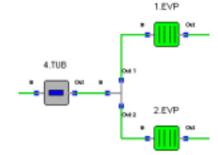


Simulation Model: Based on Geometric Properties

Condenser [Condenser.cn	d]*	The second s		_ 0
Extras S 😂 📾 📾 🕸				
eneral data] Geometric prope	tes Geore	rric properties (Inside) Geometric proper	ses (outside) Orenec, curves	
Inner flow (refrigeran	t)			
Diameter (mm)	ſ	ī		
P Non tubuler				
Wetted area [mm/]	45.6	Pinthickness s [mm]	<u> </u>	
Wetted perimeter [mm]	42.8	Fin heighth [mm]	5_n = 2	
hiner Tube height[mm] No. of longitudinelly fins n [-]	0	Finnal divided		
Thickness [mm]	0.1	Direction of inner flow	y (ny-direct)	
Pipe roughness [mm]	0.002			
		F Inputset side kest terrofer wee. Ref. side heatterrofer area [m]		
Press, loss coeff, inletition [-]	4			
	2	-		
	-			
ified				

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Components





Heat Sources



1.COM

- SLHX (Parallel Flow)
- Compressor (Controlled/Uncontrolled)
- > TXV, Orifice

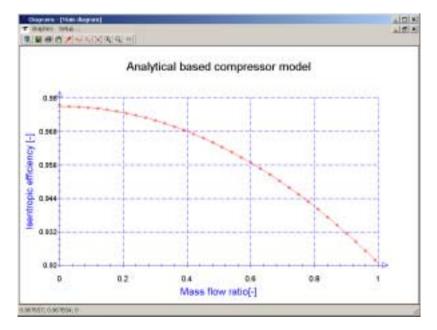
1.ACPFC [Inside]

- Condenser/Evaporator
- User-Defined External Component (using COM Interface)

Compressor

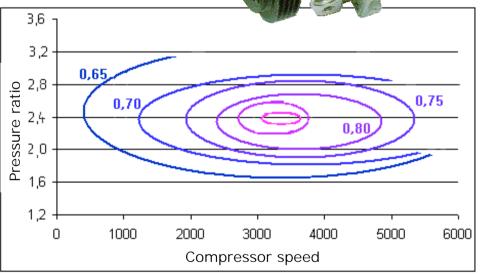
Based on Characteristic Curves

Based on an Analytical Model



I sentropic Efficiency Formula





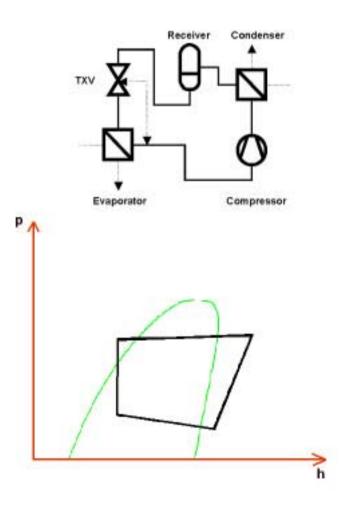
Measured Isentropic Efficiency Map

ENGINEERING CENTER STEYR

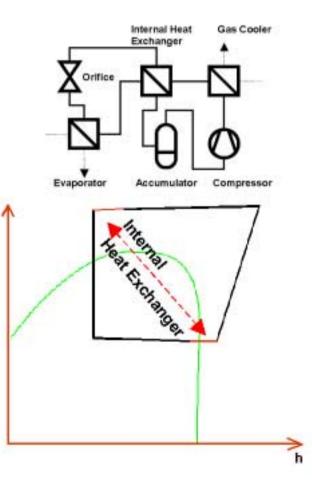
ENGINEERING

Comparison R134a-CO₂

Refrigerant R 134a



Refrigerant R 744 (CO₂)



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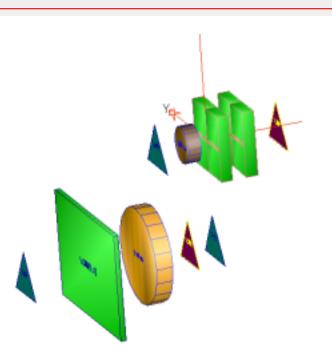
KULI AC



Application

General Data

- 1000-3500 RPM Compressor Speed
- R744 Refrigerant (CO₂)
- Orifice with Accumulator and SLHX
- Ambient Temperature 30°C, 40% Humidity
- Compressor with Outlet-Pressure Control



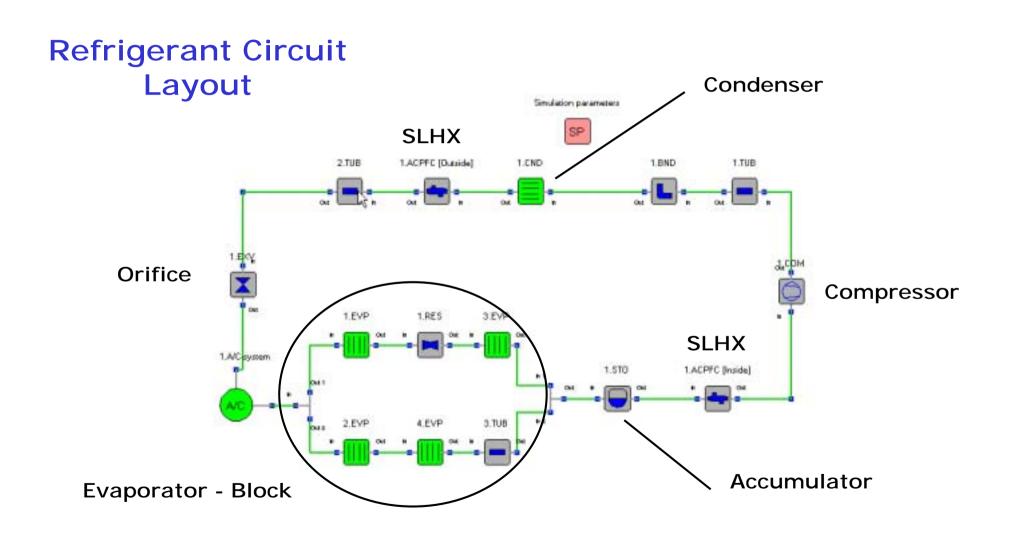


Application

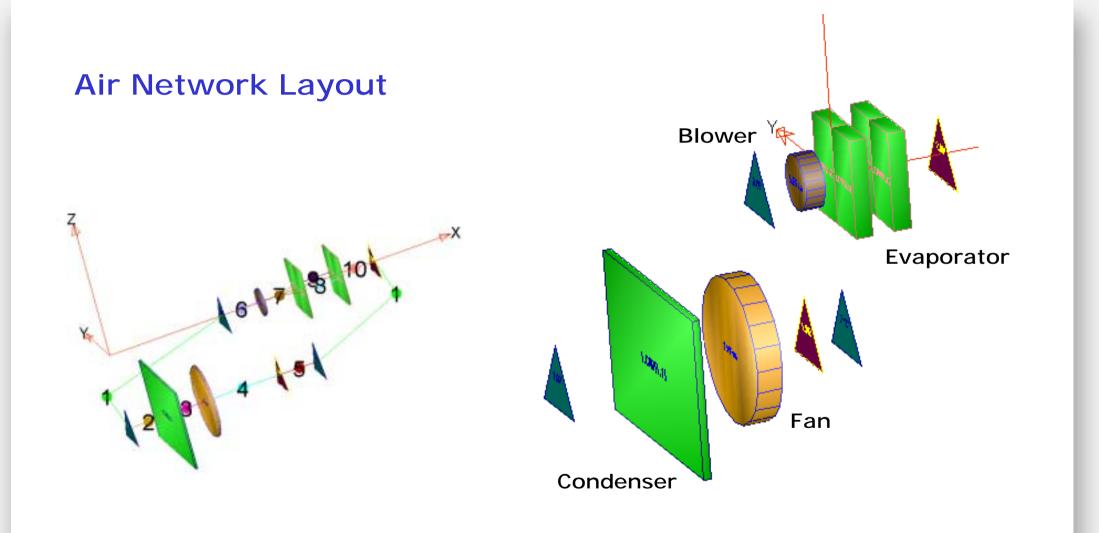
Geometric Data

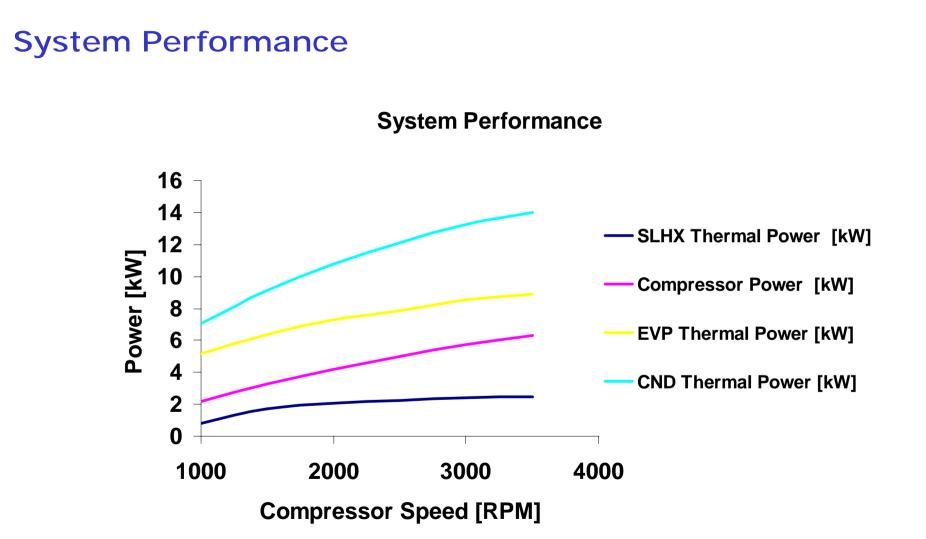
- Condenser: 600x410x20 mm, 4 Pass, 37 Tubes
- Evaporator: 250x250x90 mm, 4 Pass, 36 Tubes (Plate Type)
- Compressor Piston Displacement 30 cm³
- Orifice Expansion Device (with Accumulator)
- Pipes according their Geometry (length, diameter)

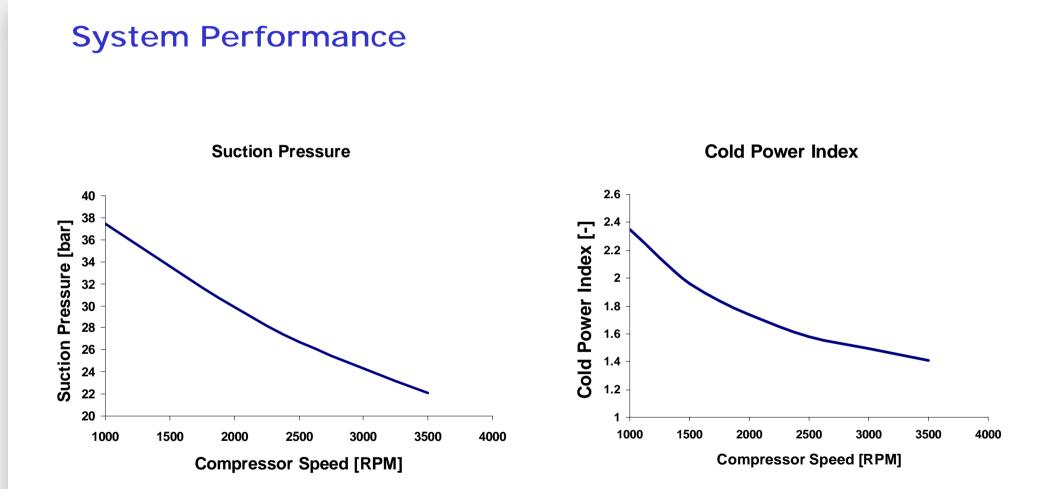
Application



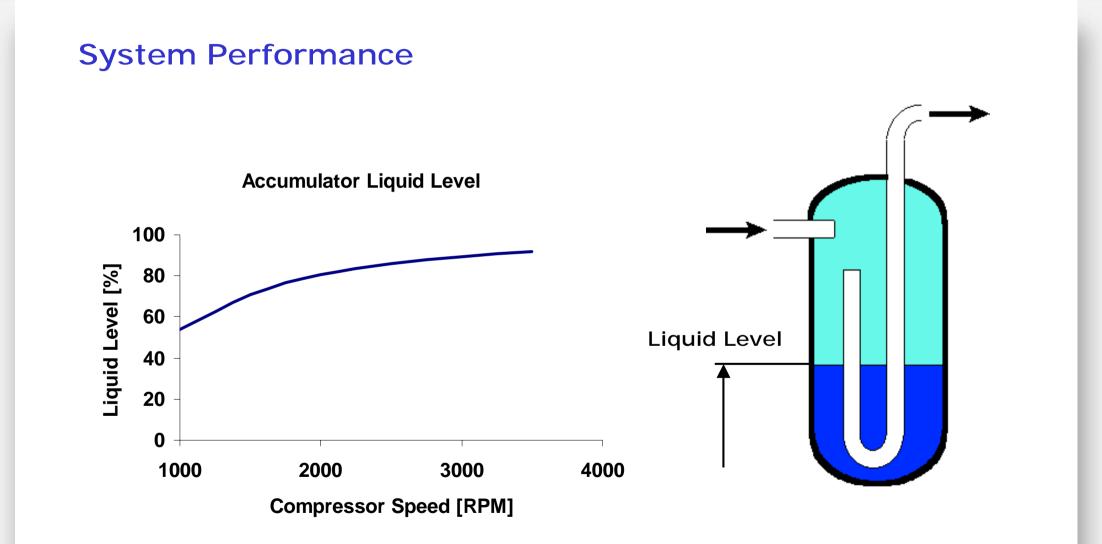
Application

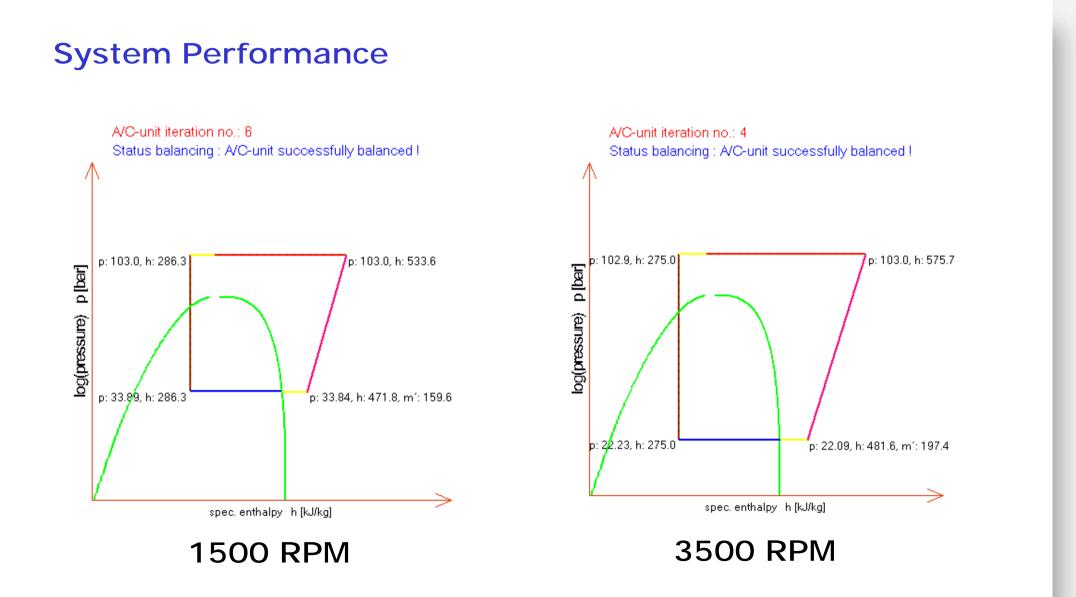






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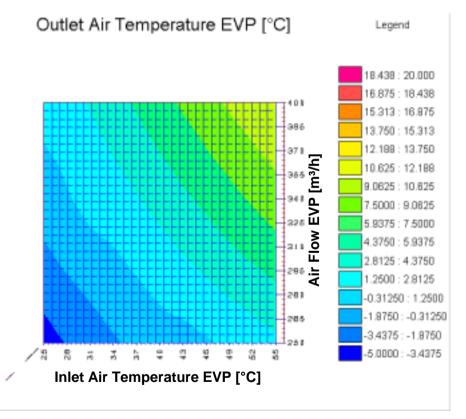


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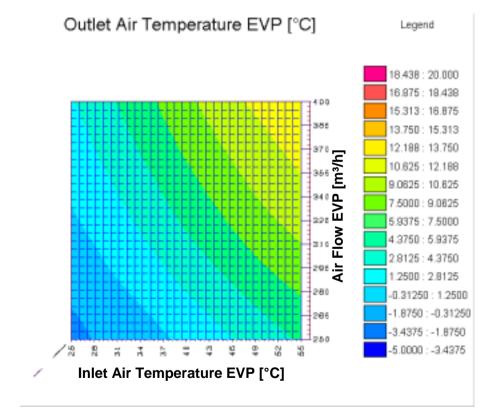
Evaporator Outlet Air Temperature

45°C Ambient Temp.



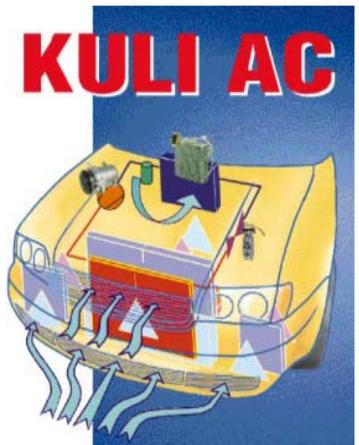
55°C Ambient Temp.

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Analysis Results

- Pressures, Temperatures (air / refrigerant)
- Refrigerant Enthalpy, Vapour Quantity
- Mass Flows (air, refrigerant)
- Heat Flows for Evaporator, Condenser, Tubes
- Condensate separation at the Evaporator
- Accumulator Liquid Level
- Required Compressor Power, ….



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Further Development

- Continue the Cooperation with TU-Graz (Institut für Wärmetechnik)
- Extended Heat Transfer Models (refrigerant/air)
- Extended Analysis Models for Pressure Loss Calculation
- More Accurate Consideration of the Lubricant Oil
- Improved Analytically-Based Compressor Model
- Graphical User Interface (GUI) Improvements



- Design of all possible Configurations of the Air and Refrigerant Flow Network
- Special Models for Receiver and Accumulator
- High Pressure Control for CO₂ Circuits
- Analysis Models supplied for R134a and R744 (CO2) Refrigerants
- Possibility to take an Inhomogeneous Velocity Distribution into Consideration (CFD-Interface)

more valueemore car





Thank You For Your Attention

Thomas Anzenberger, and the ECS-Steyr KULI Team