

# WHY coupling KULI to CFD?

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# Contents

## Why coupling KULI to CFD?

- **Introduction**
- **Data Acquisition / Workflow / Validation**
- **Scope of Work**
- **Model Setup**
- **First Results**
- **Optimization Steps**
- **Conclusions**

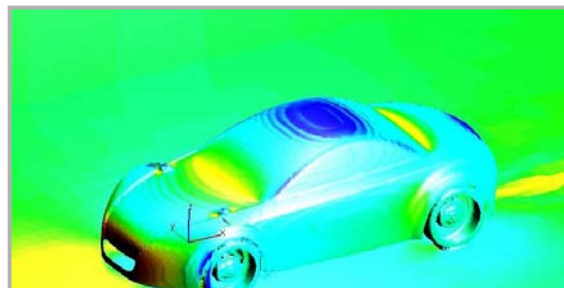
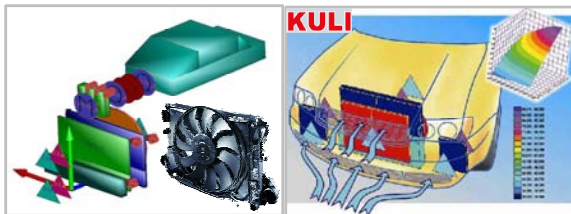
• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • Optimization • Conclusions

# Thermal Development - Simulation

## Thermal Calculation / Computational Fluid Dynamics

### Engine Cooling and HVAC System

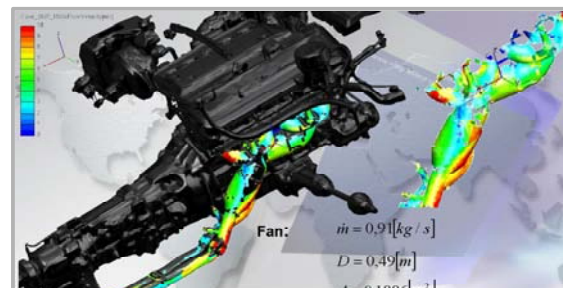
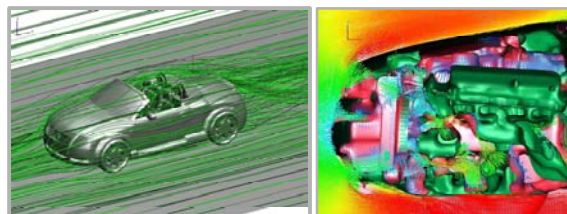
1D



simulated pressure distribution

### Aerodynamics Engine Compartment Heat Protection

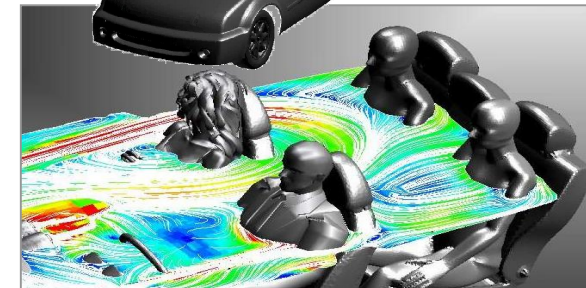
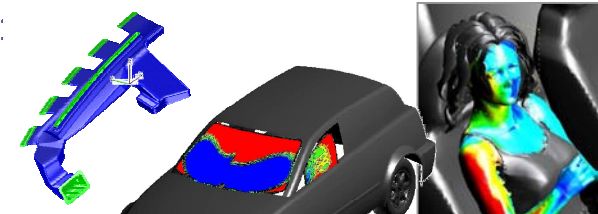
3D



simulated thermal radiation

### Channel Flow De-Icing Thermal Comfort

3D

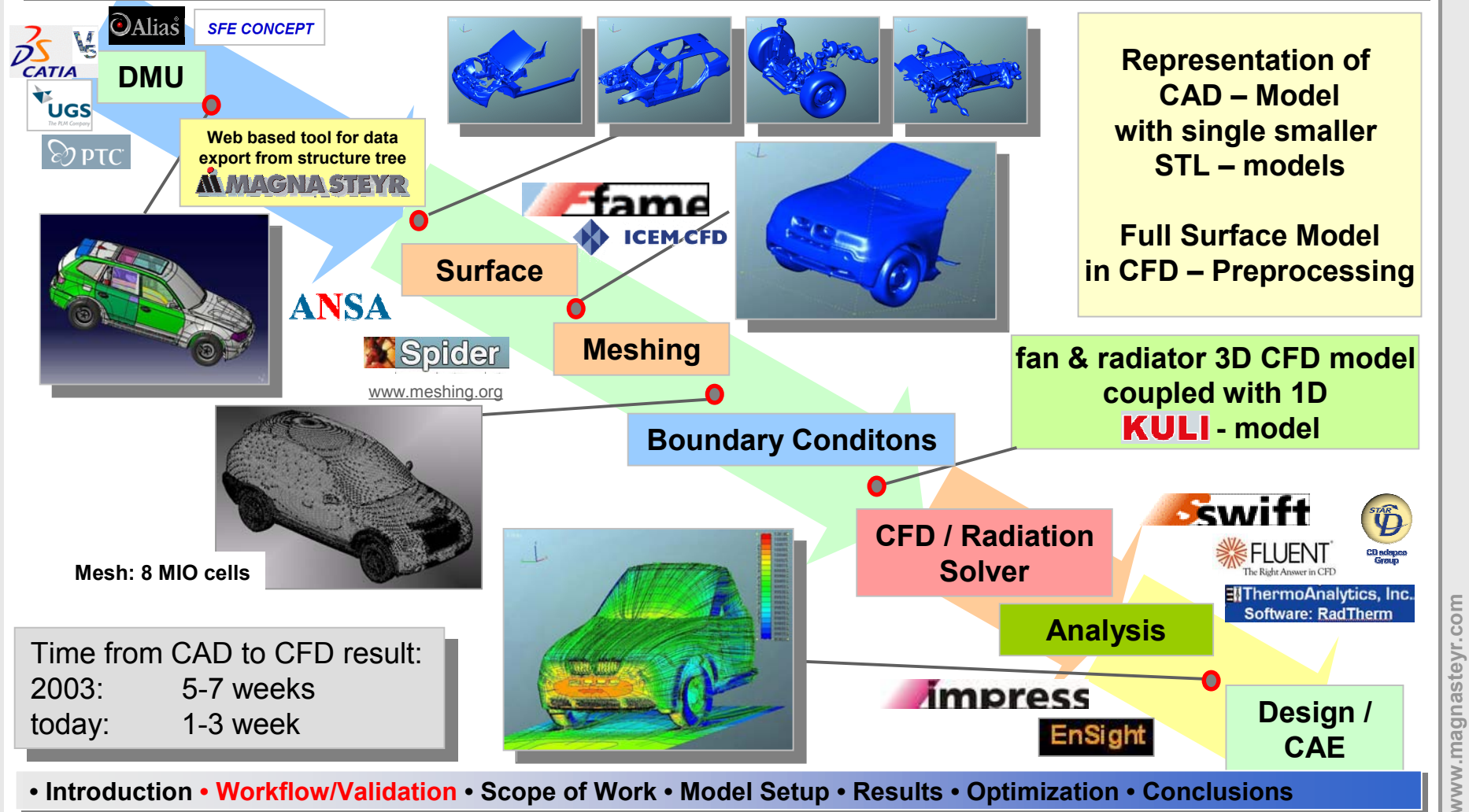


simulated A/C comfort

• **Introduction** • Workflow/Validation • Scope of Work • Model Setup • Results • Optimization • Conclusions

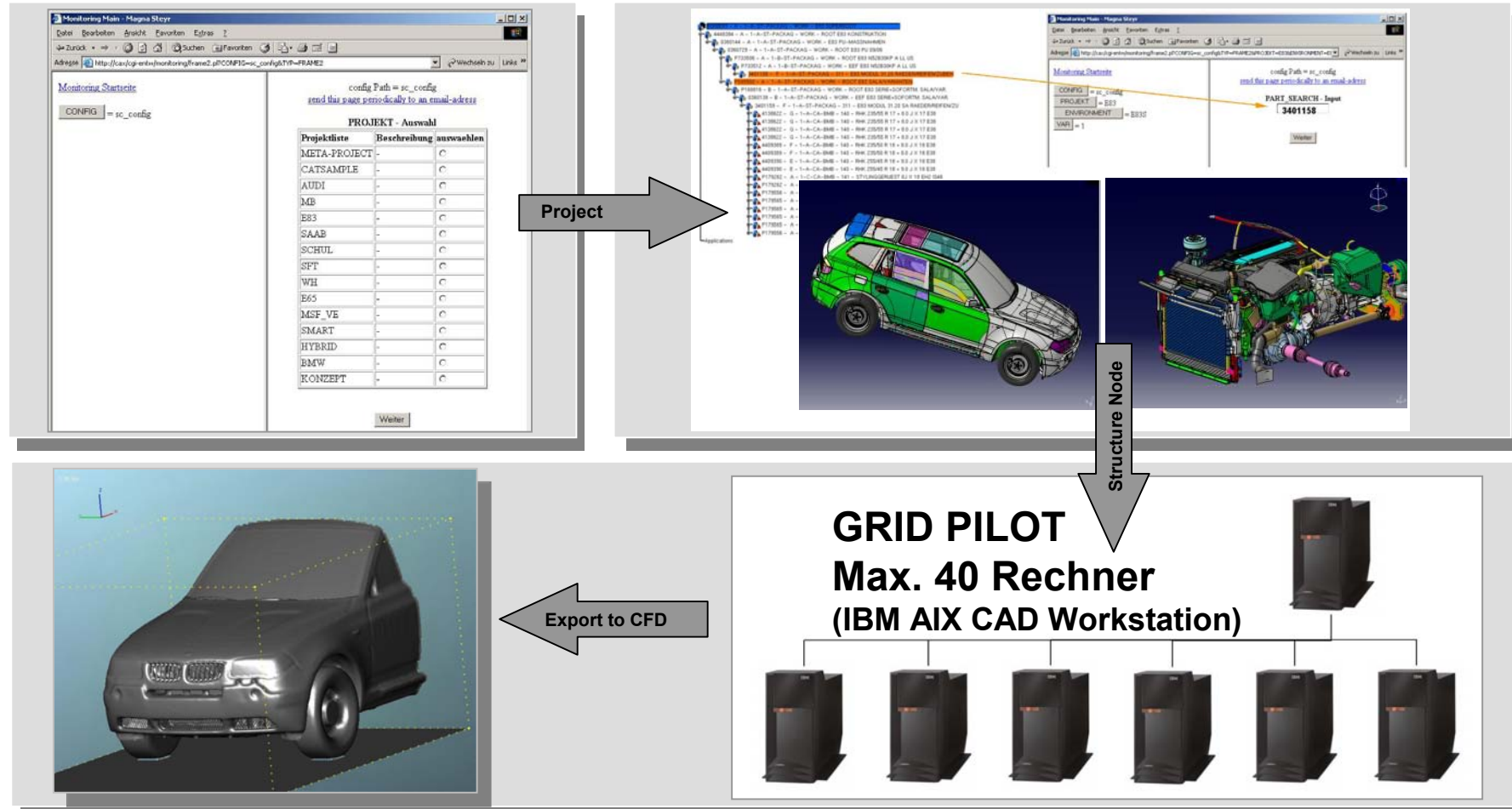
# Data - Acquisition

## Workflow CAD – CFD (plus 1D coupling)



# Data - Acquisition

## Web based tool for data export from vehicle structure tree



• Introduction • **Workflow/Validation** • Scope of Work • Model Setup • Results • Optimization • Conclusions

# Tools and Validation

## Hardware/Software Tools and Virtual Methods

### HARDWARE:

- Linux Cluster (48 CPUs, 100 GB MEM, 64bit)
- HP Itanium II (2 CPUs, 12 GB MEM, 64bit)
- HP PC WS (2 CPUs, 4 GB MEM, 32bit)
- Opteron PC WS (2 CPUs, 8 GB MEM, 64bit)

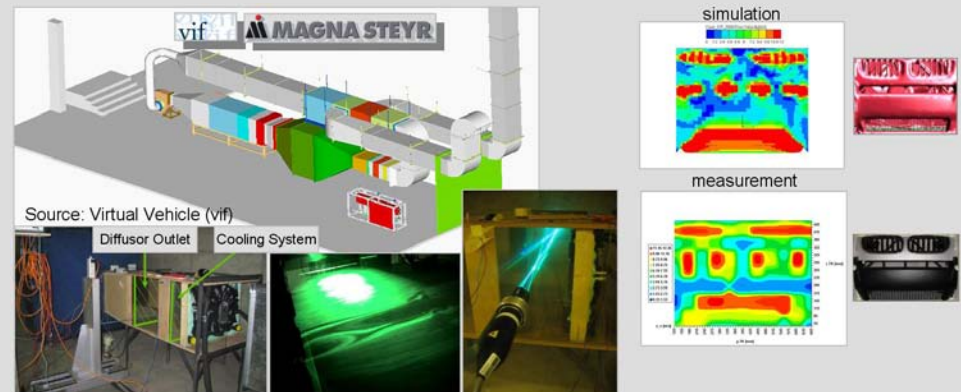
### SOFTWARE:

- Catia v5 DMU Navigator
- SWIFT (StarCD, FLUENT)
- KULI
- RadTherm / INKA / TILL
- Spider, CFDWM



### MODEL VALIDATION:

- Component Wind Tunnel with Laser Doppler Anemometry (LDA) for Cooling Components
- Thermal material data measurements

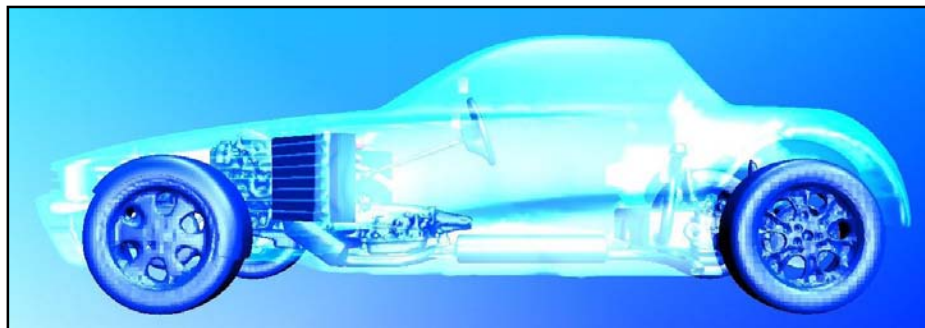
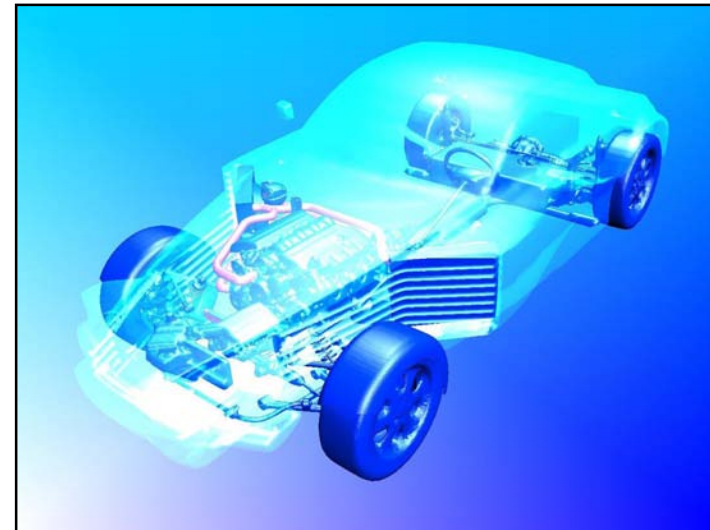
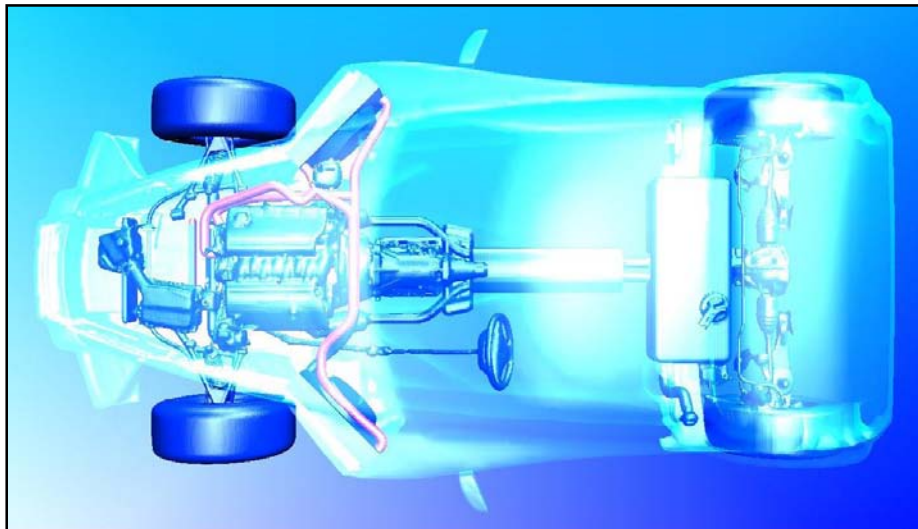


- Testing of cooling system performance
- Validation of simulation models (Radiator, Fan, ...)
- Detection of emissivity / absorptivity coefficients

• Introduction • **Workflow/Validation** • Scope of Work • Model Setup • Results • Optimization • Conclusions

## Scope of Work

### Concept Car: Feasibility Study



Design a cooling system for

- V8 Engine Power: 295 kW
- Maximum Speed: 240 kmph

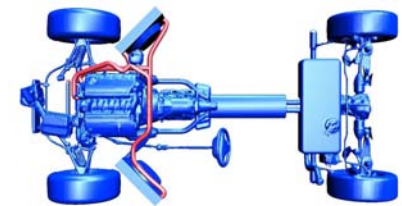
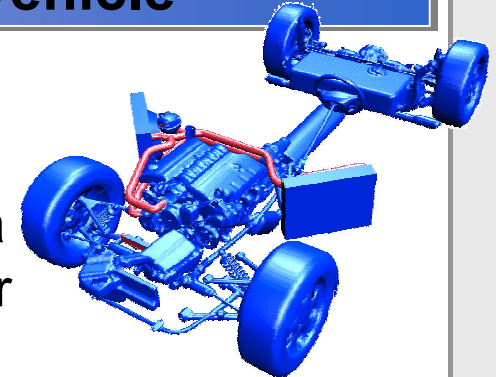
with radiators on the sides

• Introduction • Workflow/Validation • **Scope of Work** • Model Setup • Results • Optimization • Conclusions

# Cooling System Behavior

## Problems expected due to configuration of the vehicle

- Too less airflow through the radiator to guarantee adequate efficiency of the cooling system.
- In order to reach an adequate cooling of the system, a cp-pressure of at least 0.5 has to be reached, together with an air mass flow through the radiator of at least 1.5 kg/sec.
- possible backflows out of the underhood due to negative pressure differences between the inside and the outside of the radiator and a low-pressure field behind the tire at high speeds which would negatively influence the flow field in front of the radiator.
- Very high thermal loads to reach top-speed due to a high drag-coefficient.



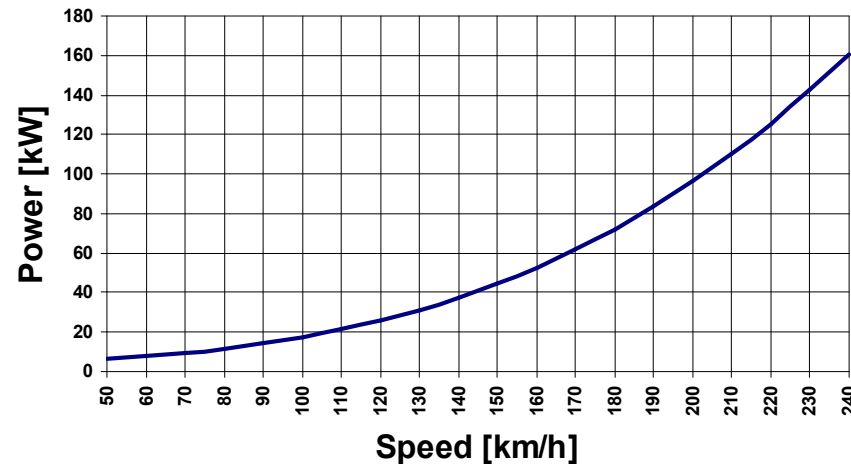
# Cooling System Behavior

## Boundary conditions of the cooling system

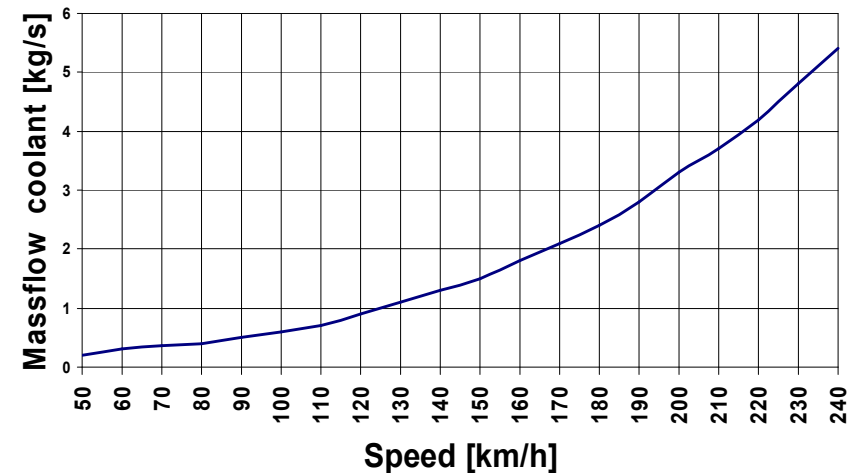
In order to calculate the efficiency of the cooling-system, the following radiator-sizes were chosen:

- radiators: two times, each with an area of at least 16 dm<sup>2</sup>
- condenser: area at least 29 dm<sup>2</sup>

heat-quantities dependent  
on the driving speed



necessary mass-flow  
of the cooling fluid dependent  
on the driving velocity



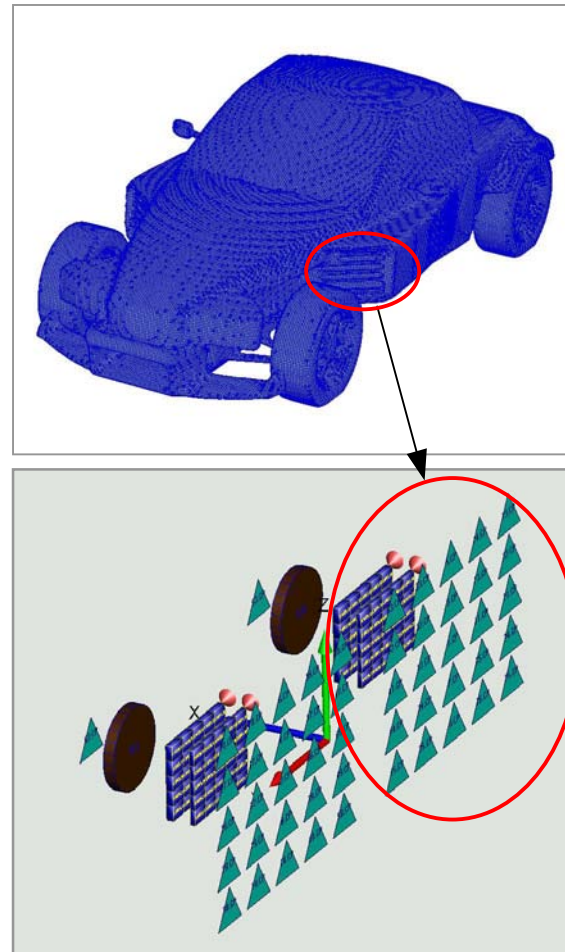
• Introduction • Workflow/Validation • **Scope of Work** • Model Setup • Results • Optimization • Conclusions

## Model Setup

### Simulation Model, Load Cases

#### Load Cases

- Hill, 60 km/h without trailer, 35°C ambient temperature
- Hill, 60 km/h with trailer, 35°C ambient temperature
- Hill, 90 km/h without trailer, 35°C ambient temperature
- Hill, 90 km/h with trailer, 35°C ambient temperature
- Vmax without trailer, 240 km/h, 45°C ambient temperature
- Vmax with trailer, 180 km/h, 45°C ambient temperature



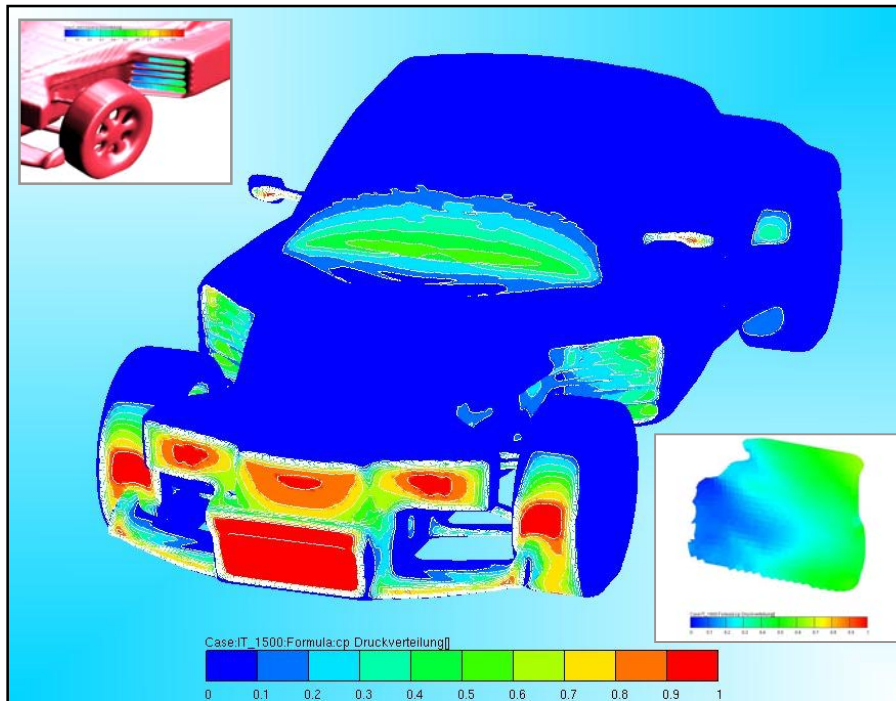
**Coupling of  
CFD  
and KULI  
via a  
 $c_p$ -Matrix**

• Introduction • Workflow/Validation • Scope of Work • **Model Setup** • Results • Optimization • Conclusions

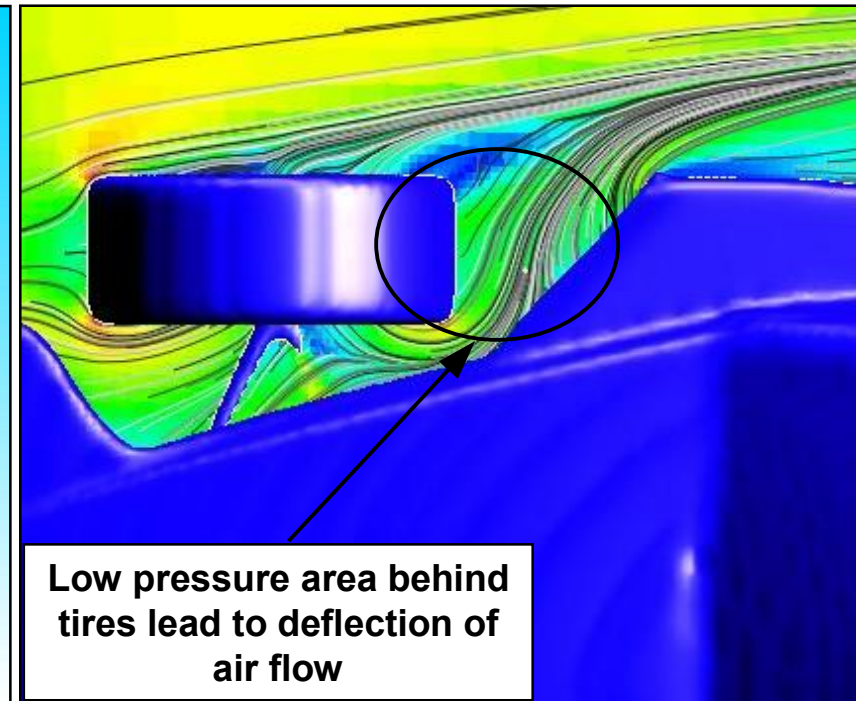
## CFD Results Base Version (A)

### $c_p$ distribution and streamlines

$c_p$ -distribution over vehicle, 3D  
calculation



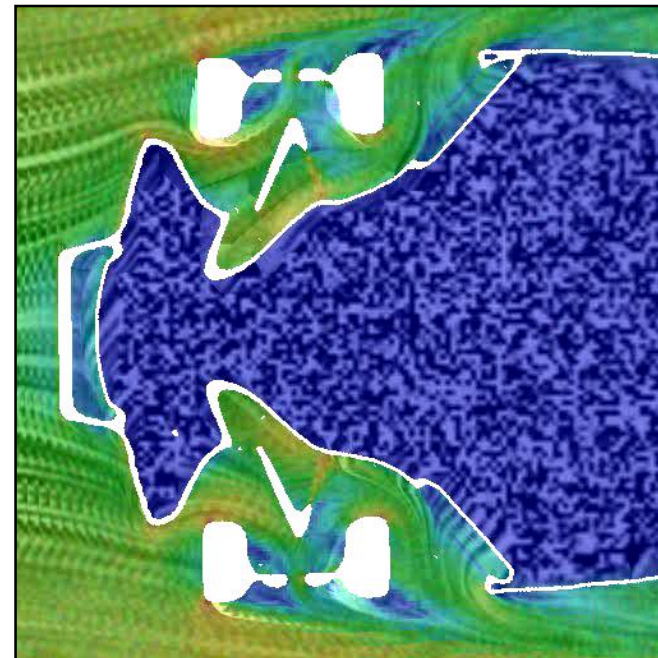
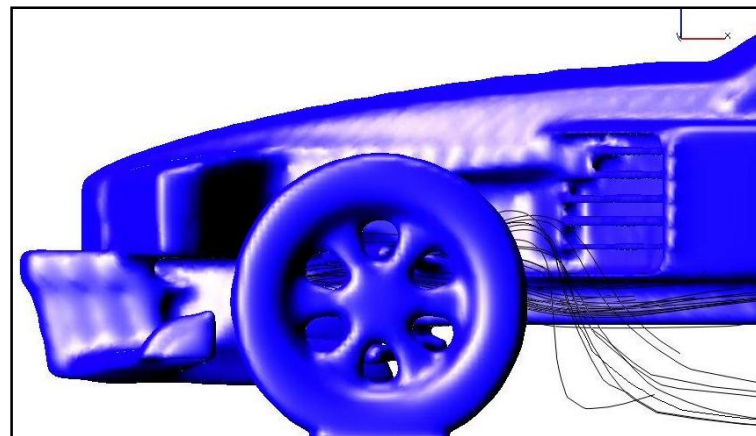
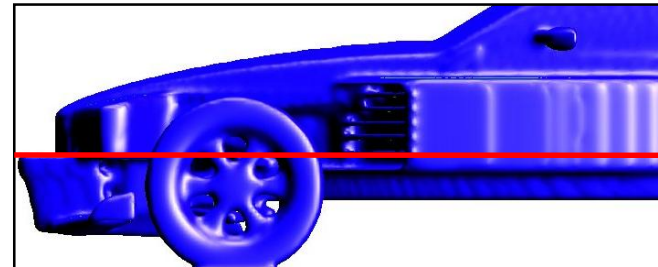
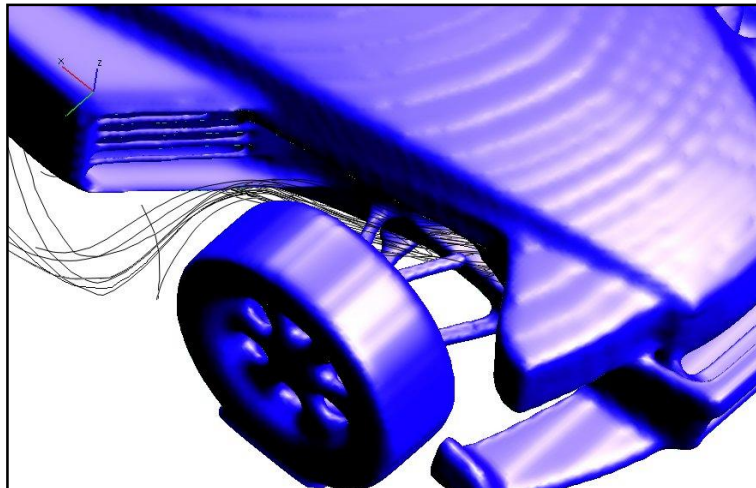
stream lines showing the air-  
flow behind the tire



• Introduction • Workflow/Validation • Scope of Work • Model Setup • **Results** • Optimization • Conclusions

## CFD Results Base Version (A)

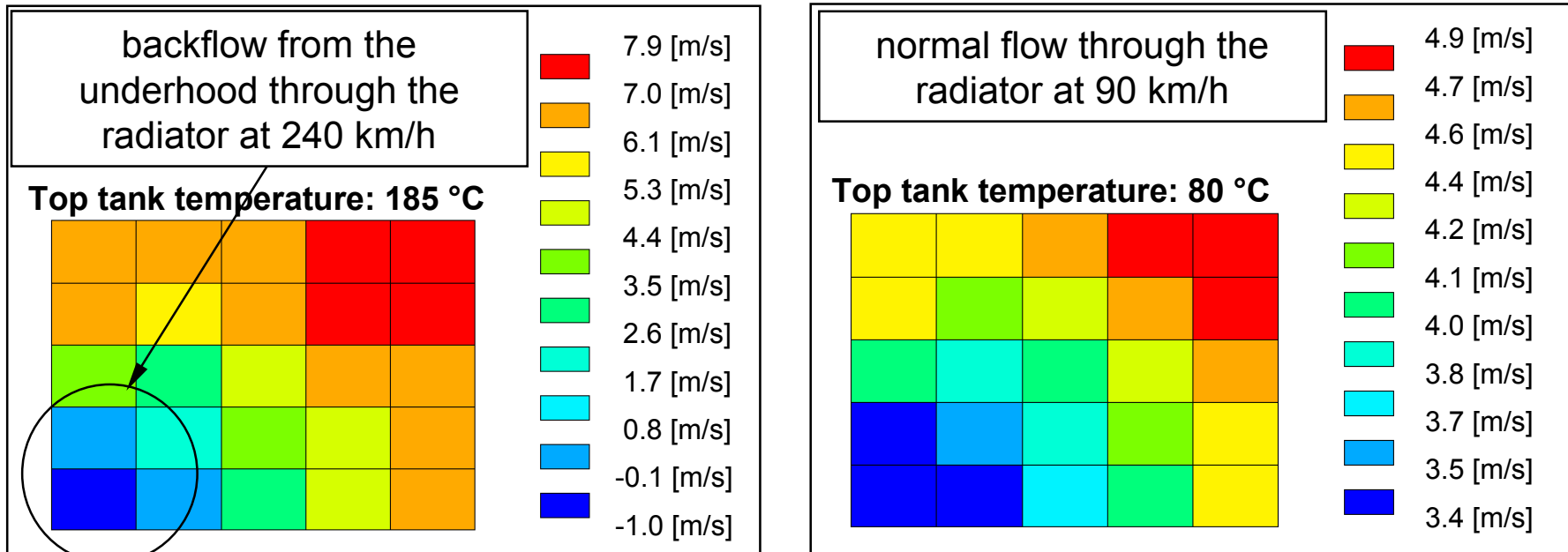
### Streamlines and velocities

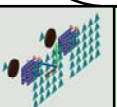


• Introduction • Workflow/Validation • Scope of Work • Model Setup • **Results** • Optimization • Conclusions

# KULI Results Base Version (A)

## Air Massflow



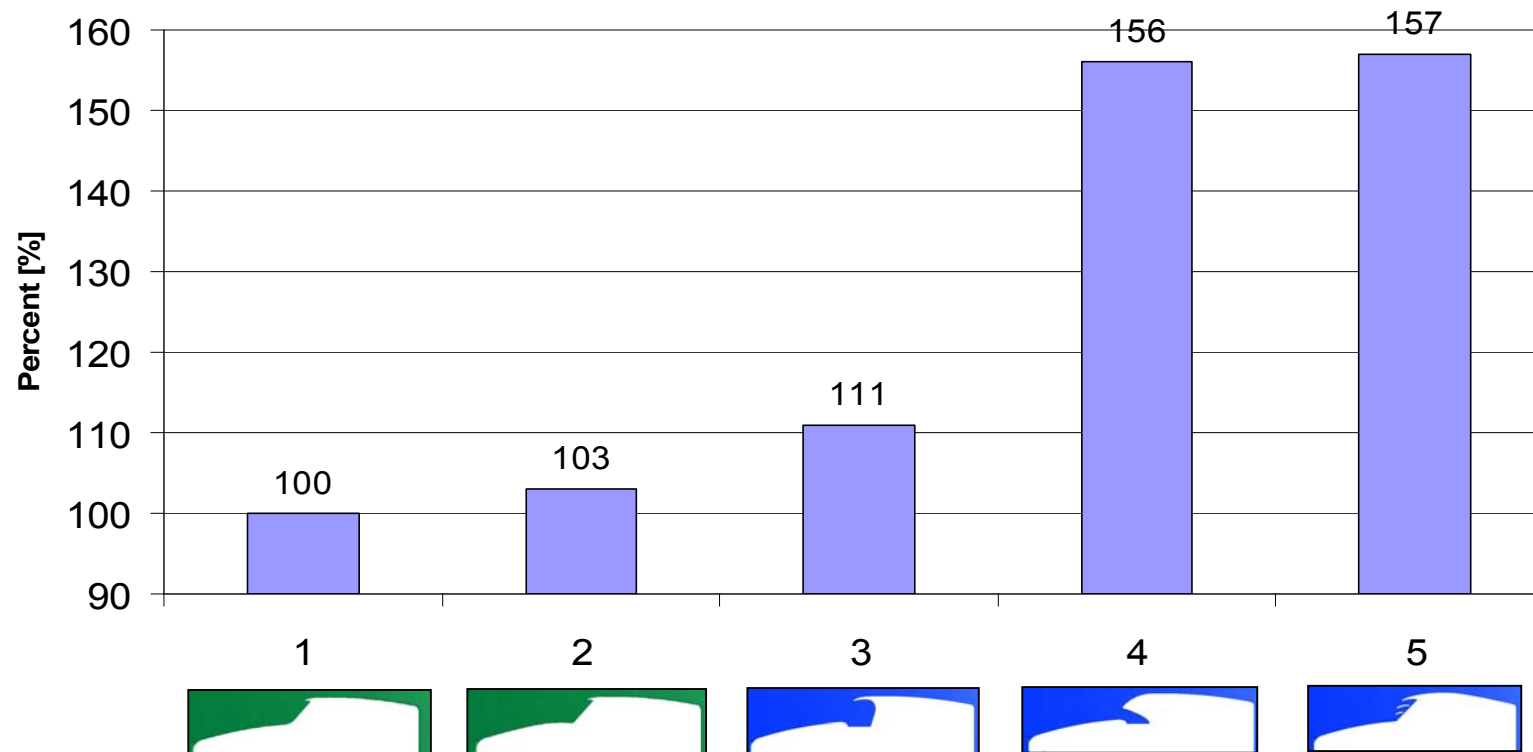
|  |                                   |                           |  |  |   |  |   |   |   |
|--|-----------------------------------|---------------------------|--|--|---|--|---|---|---|
|  <div>KULI Simulation</div> |                                   | 450W Electrical Fan       |  |  |   |  |   |   |   |
|  |                                   | responsible for modelling | Idle 1000 RPM<br>outside: 45°C<br>Limit: 115°C | Hill 7% with<br>60 km/h<br>outside: 35°C<br>Limit: 105°C | Hill 7% with<br>60 km/h and<br>trailer<br>outside: 35°C<br>Limit: 115°C | Hill 6% with<br>60 km/h<br>outside: 35°C<br>Limit: 105°C | Hill 6% with<br>90 km/h and<br>trailer<br>outside: 35°C<br>Limit: 115°C | Vmax 240<br>km/h<br>outside: 45°C<br>Limit: 113°C | Vmax with<br>trailer 180<br>km/h<br>outside: 45°C<br>Limit: 115°C |
| A  | Base version without any measures | Christian Kussmann        | max. heat<br>54 kW                             | 57.1   | 67.4  | 65.3   | 78.9  | 181.2   | 118.1   |

• Introduction • Workflow/Validation • Scope of Work • Model Setup • **Results** • Optimization • Conclusions

# Optimization

## 2D study

In order to find an optimised air inlet geometry, different situations have been calculated in 2D. The optimal variant concerning  $c_p$  was then chosen to be modelled in 3D.

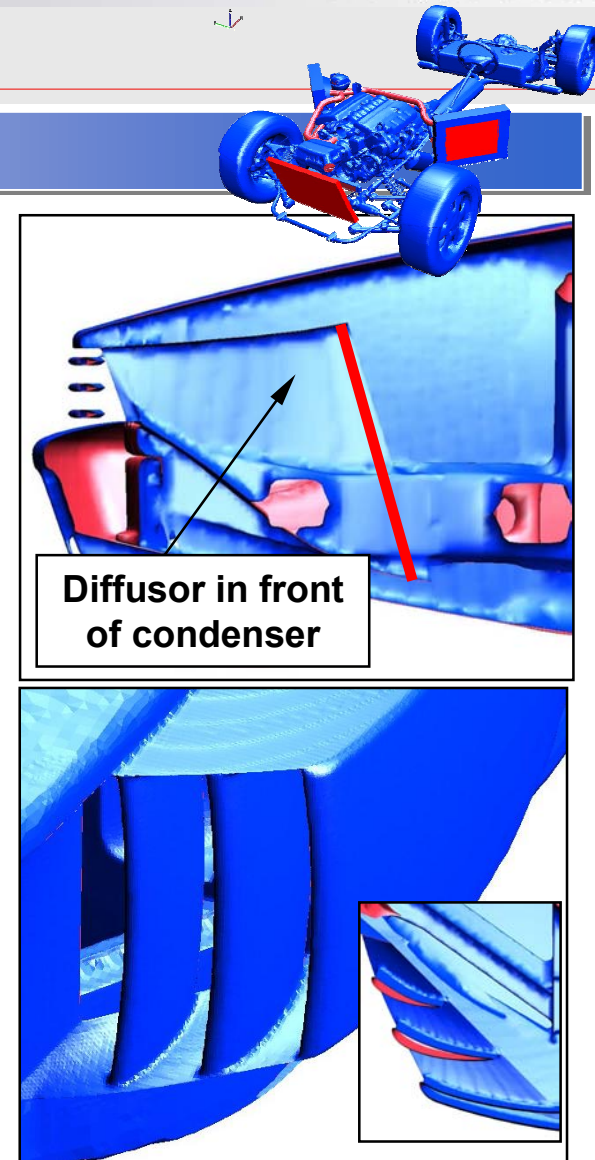


• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

## Optimization 2<sup>nd</sup> Version (C)

### Aerodynamic Improvements

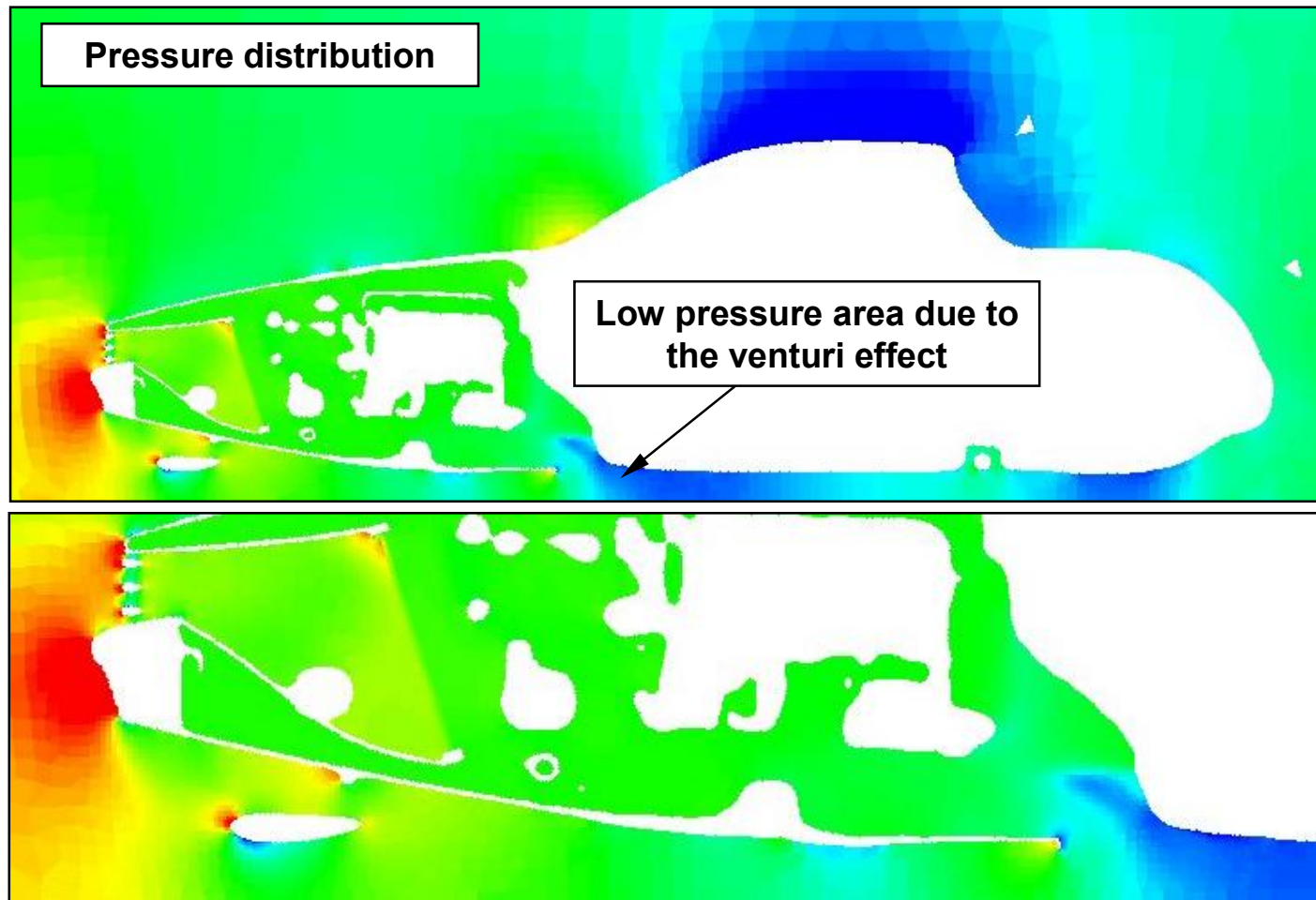
- the underbody has been shaped like a venturi nozzle to generate a low pressure area below the car in order to improve the cp-difference. In the centre of this area, the air could leave the underhood and join the surrounding air.
- The climate condenser has been moved to the front of the car in order to function like an air restrictor to prevent high pressures to be produced in the underhood. To intensify this effect, the air duct in front of the condenser has been shaped to function as a diffuser. The second main effect of the movement of the condenser is the unaffected air flow through the radiator, as there is no resistor in front of it anymore.
- vertical vanes have been designed in order to guide the air directly to the surface of the radiator. The fins of this guidance system have been shaped to form a diffuser system as well.



• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

## CFD Results Optimization 2<sup>nd</sup> Version (C)

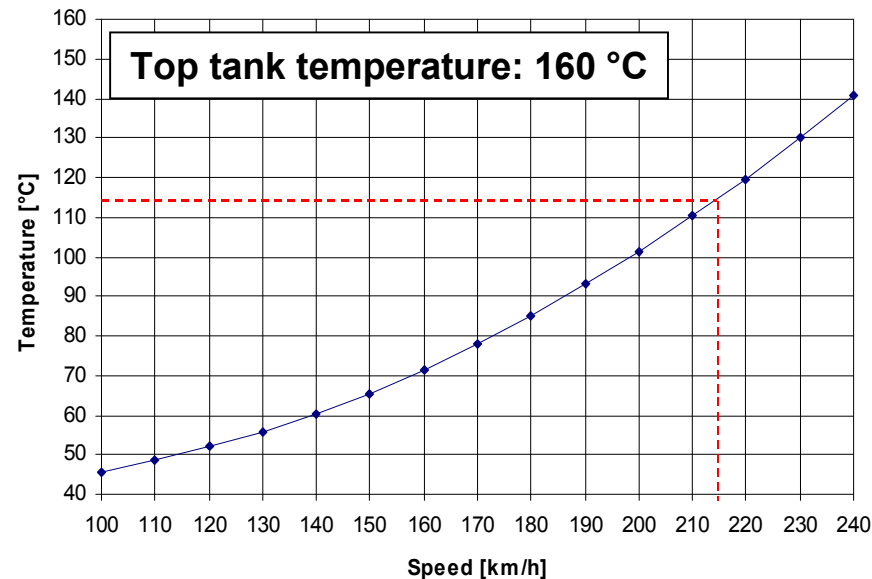
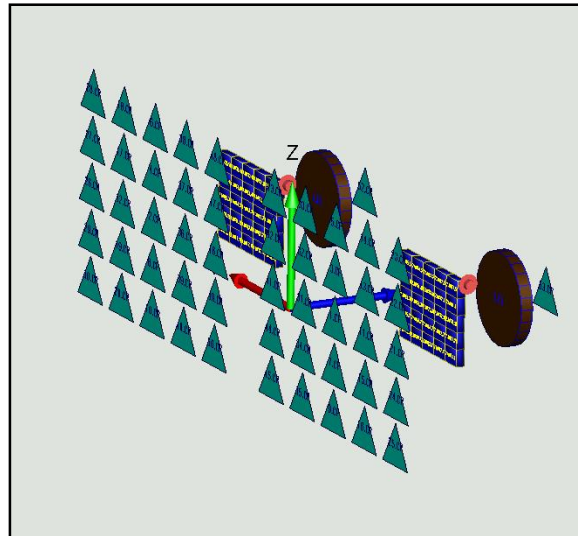
### Pressure Distribution



• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

# KULI Results Optimization 2<sup>nd</sup> Version (C)

## Cooling Performance (Top Tank Temperature)



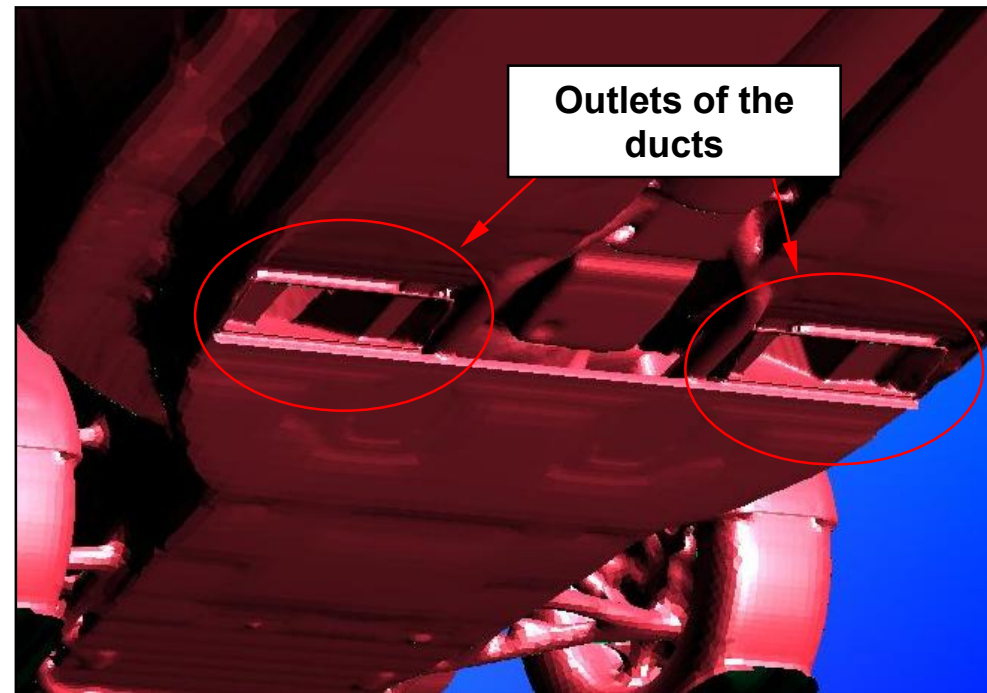
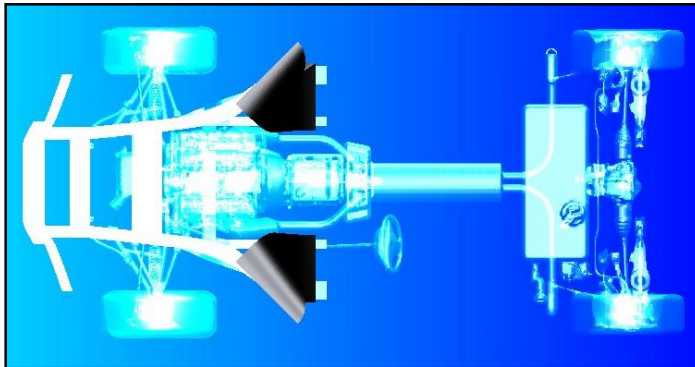
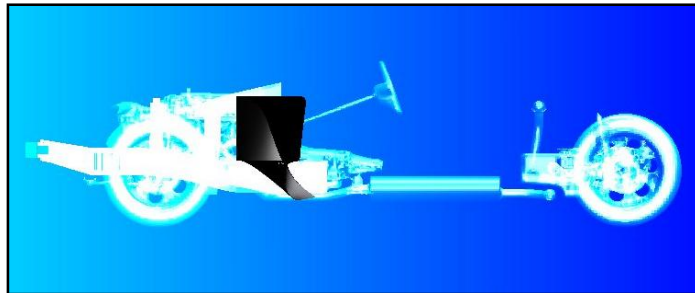
| KULI Simulation |   | 450W Electrical Fan       |  |  |   |  |   |   |   |
|-----------------|---|---------------------------|--|--|---|--|---|---|---|
|                 |   | responsible for modelling | Idle 1000 RPM<br>outside: 45°C<br>Limit: 115°C | Hill 7% with<br>60 km/h<br>outside: 35°C<br>Limit: 105°C | Hill 7% with<br>60 km/h and<br>trailer<br>outside: 35°C<br>Limit: 115°C | Hill 6% with<br>60 km/h<br>outside: 35°C<br>Limit: 105°C | Hill 6% with<br>90 km/h and<br>trailer<br>outside: 35°C<br>Limit: 115°C | Vmax 240<br>km/h<br>outside: 45°C<br>Limit: 113°C | Vmax with<br>trailer 180<br>km/h<br>outside: 45°C<br>Limit: 115°C |
| A               | Base version without any measures                                       | Christian Kussmann        | max. heat<br>54 kW                             | 57.1   | 67.4  | 65.3   | 78.9  | 181.2   | 118.1   |
| C               | New underhood with diffusor, condenser in front and vertical inlet fins | Christian Kussmann        | max. heat<br>64 kW                             | 50.7   | 59.6  | 58.3   | 70.1  | 159.9   | 106.3   |

• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

## Optimization 3<sup>rd</sup> Version (R)

### Geometry Improvements

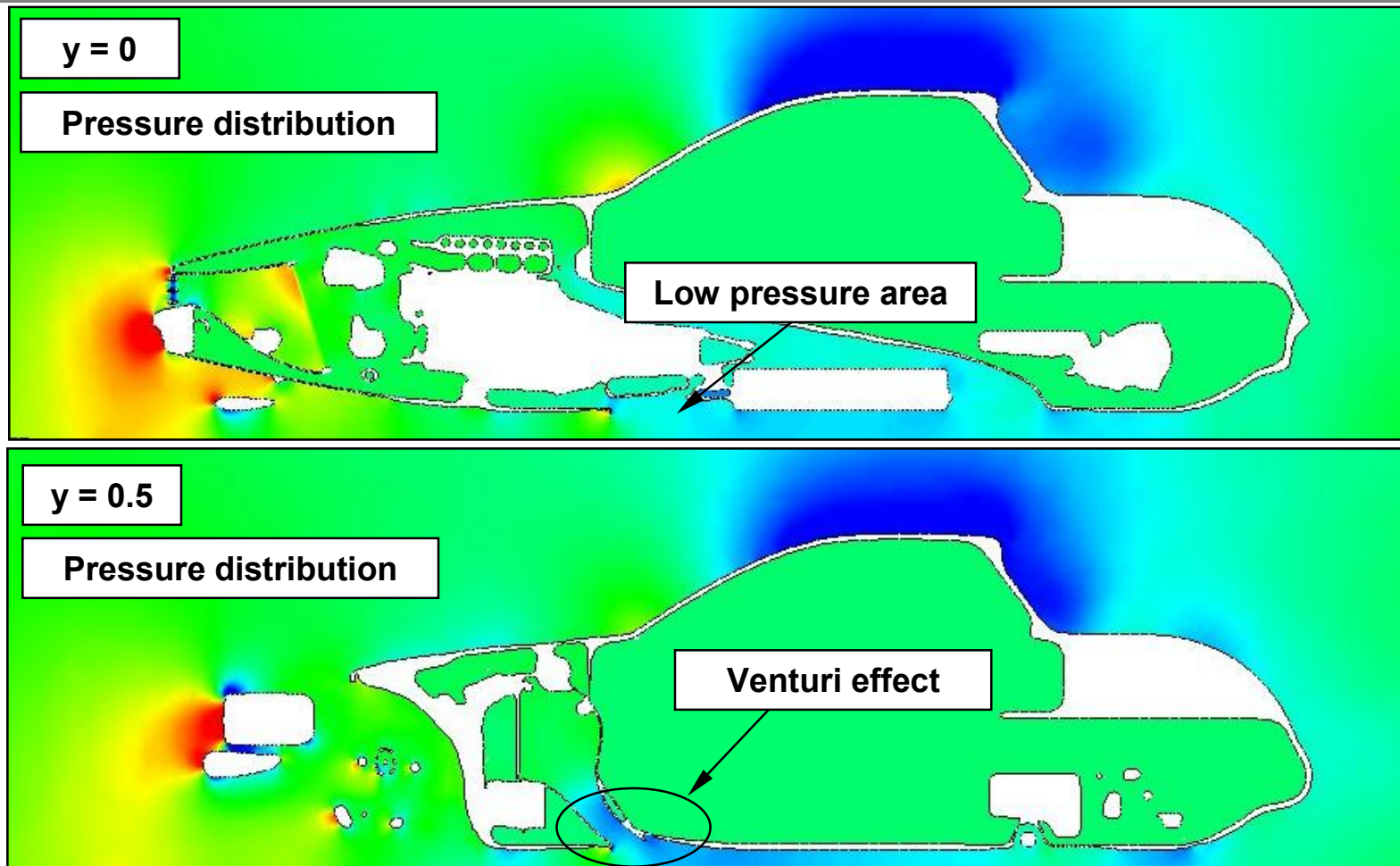
- In order to maximize the cp-difference between radiator front and cooling-package outlet, a duct was implemented. This construction connects the radiator directly with the low pressure area at the underbody of the car.



• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

## CFD Results Optimization 3<sup>rd</sup> Version (R)

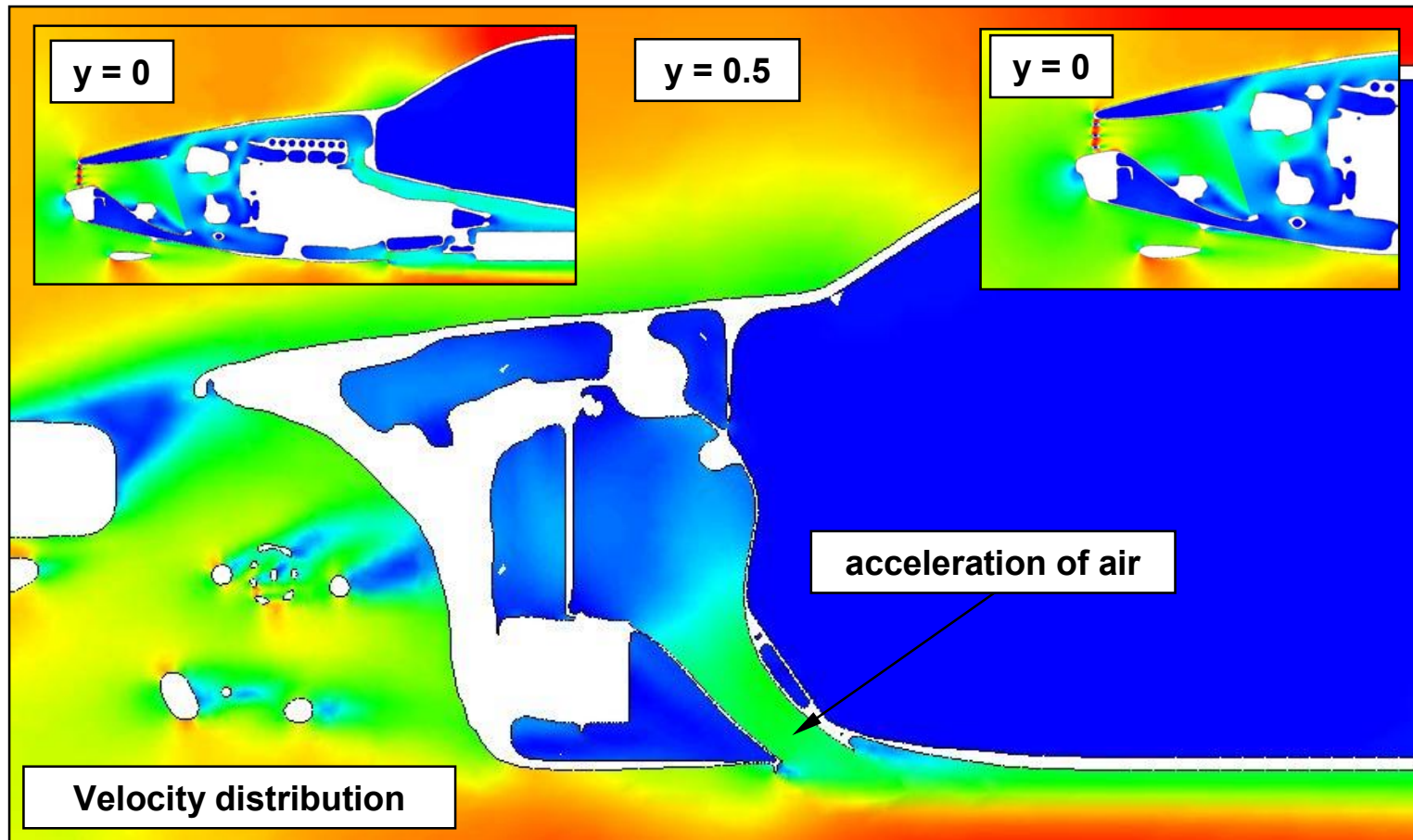
### Pressure Distribution



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## CFD Results Optimization 3<sup>rd</sup> Version (R)

### Velocity Distribution

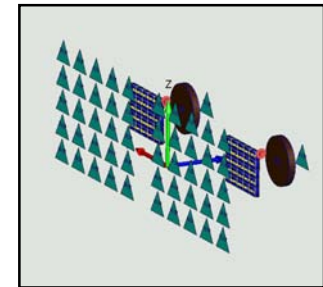
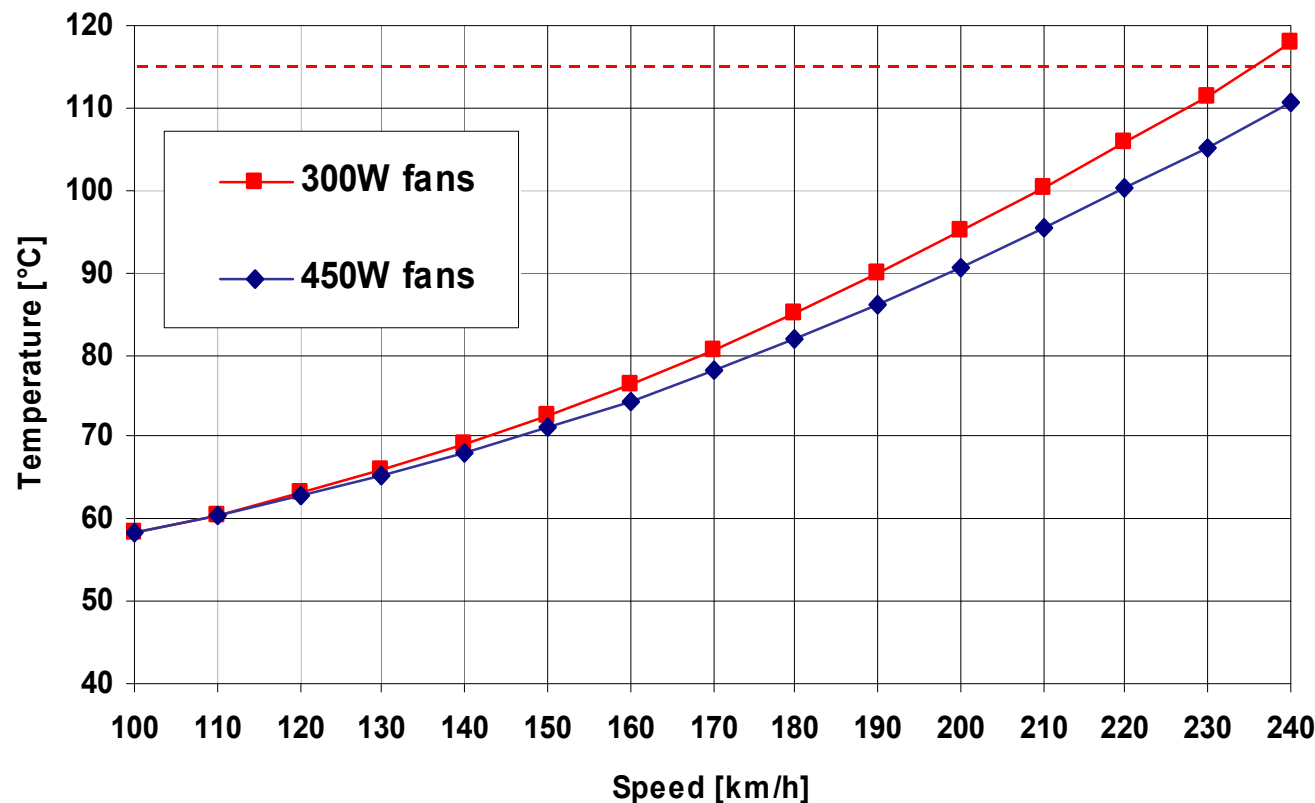


• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

# KULI Results Optimization 3<sup>rd</sup> Version (R)

## Cooling Performance (Top Tank Temperature)

1D simulation results calculated with 450 W and 300 W fans, 45°C ambient temperature and a radiator with 32mm depth.

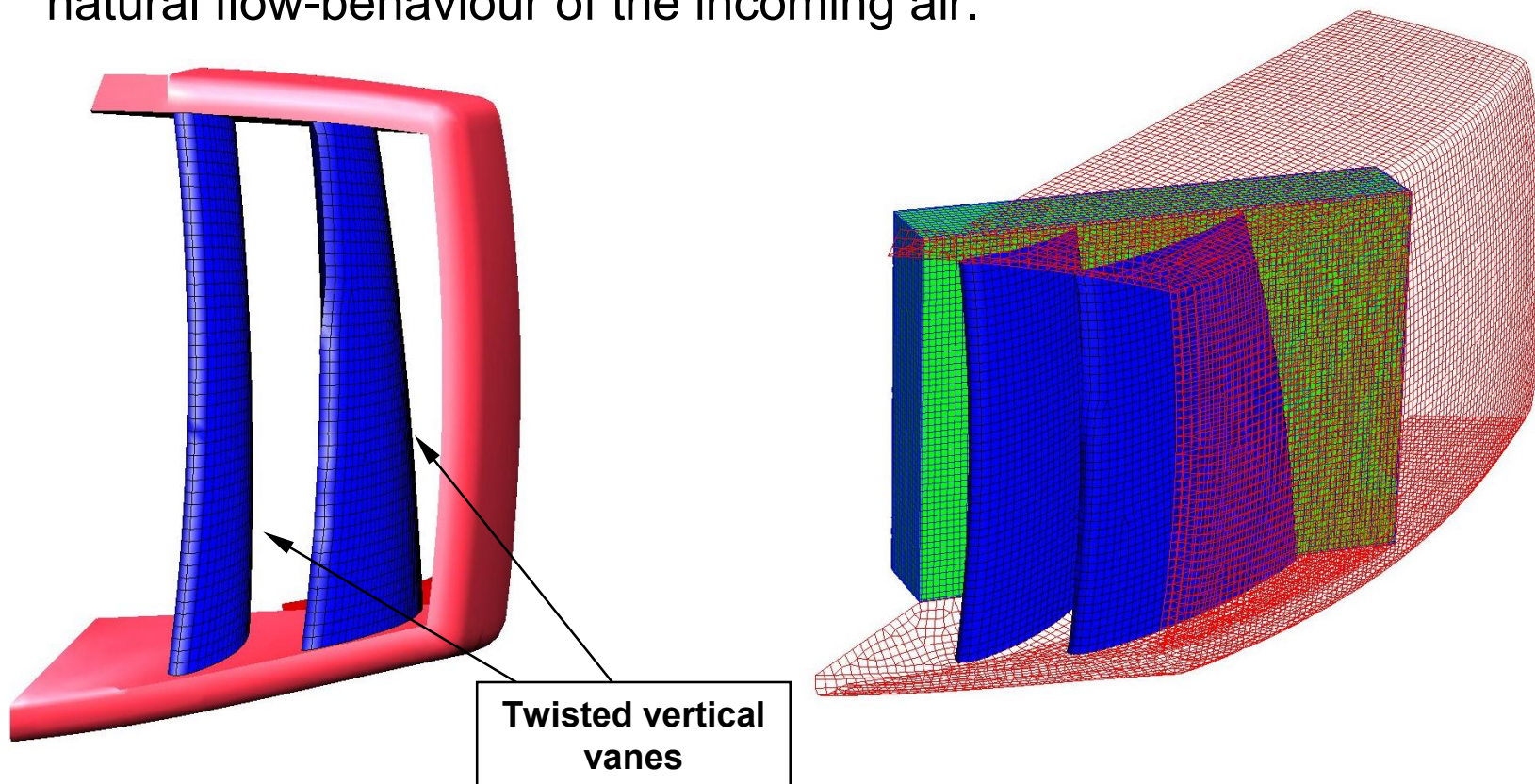


• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

## Optimization 4<sup>th</sup> Version (S)

### Geometry Improvements

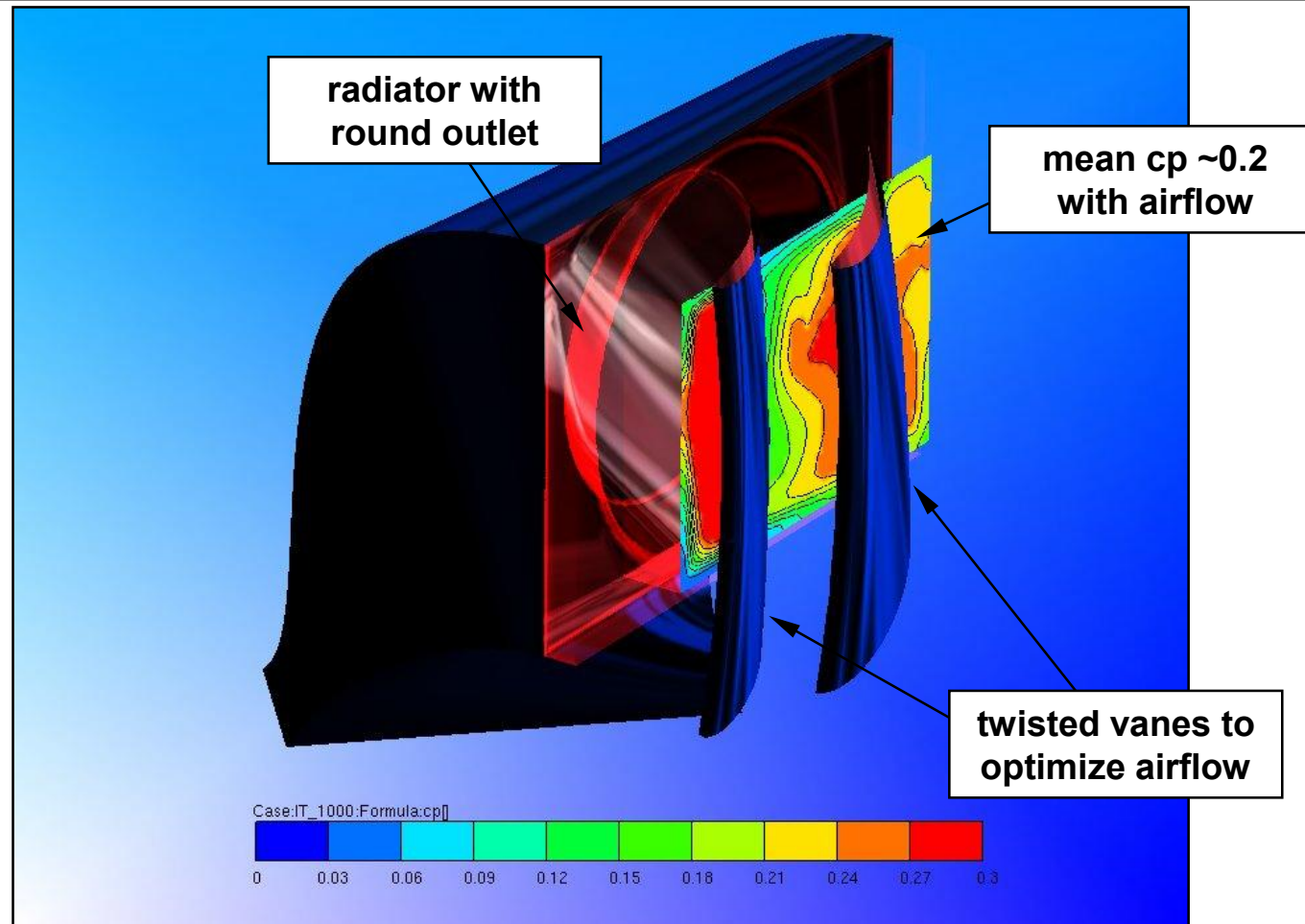
- A further improvement was to turn the vertical vanes regarding to the natural flow-behaviour of the incoming air.



• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

## CFD Results Optimization 4<sup>th</sup> Version (S)

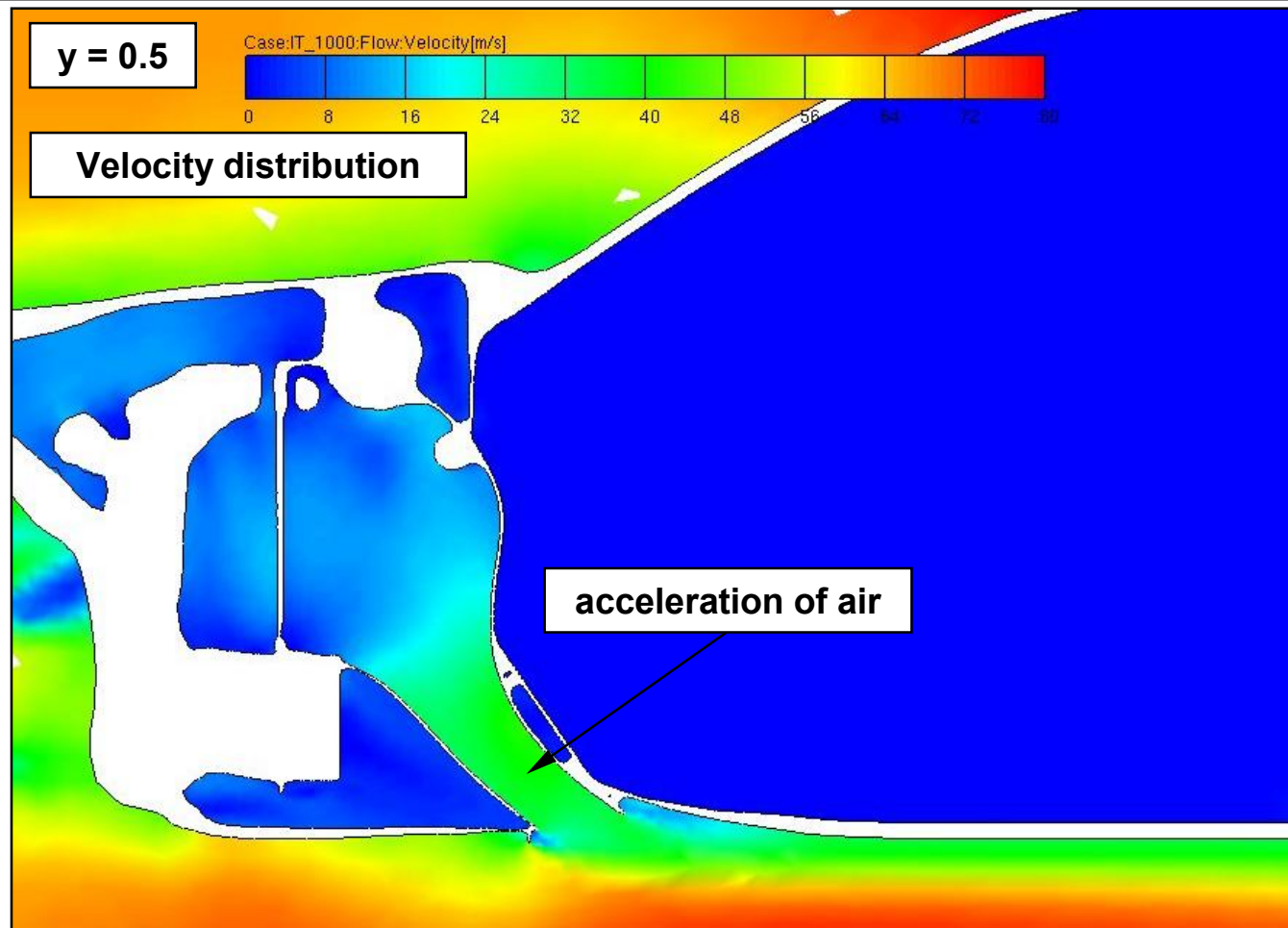
### $c_p$ Distribution



• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

## CFD Results Optimization 4<sup>th</sup> Version (S)

### Velocity Distribution

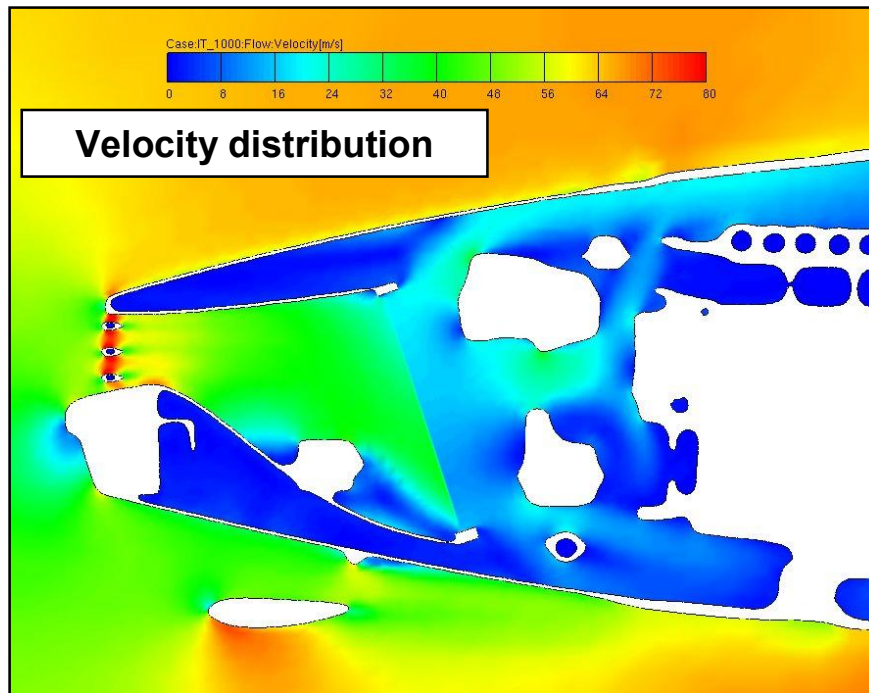


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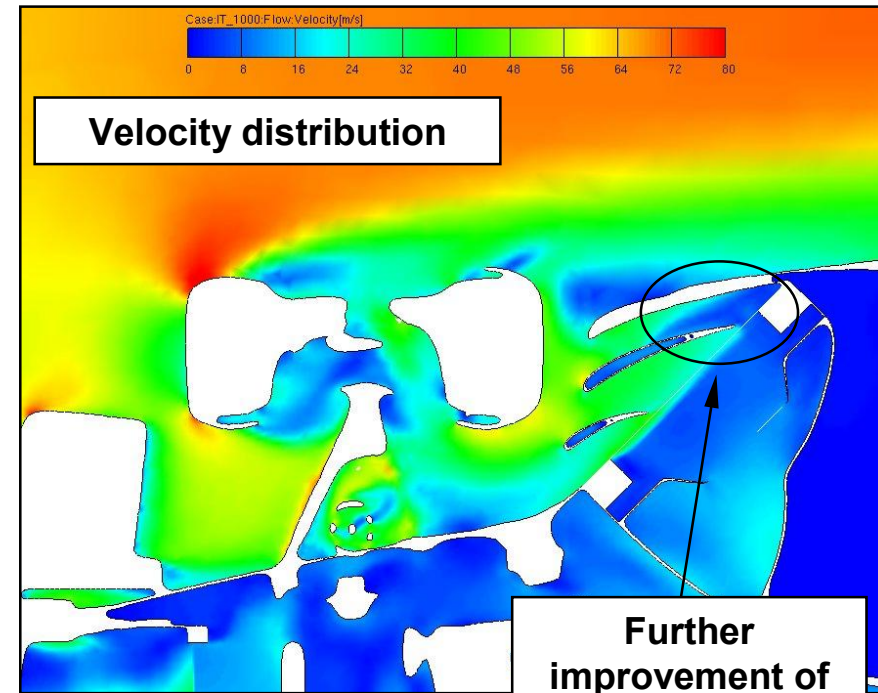
## CFD Results Optimization 4<sup>th</sup> Version (S)

### Velocity Distribution

$y = 0$



$z = 0.9$

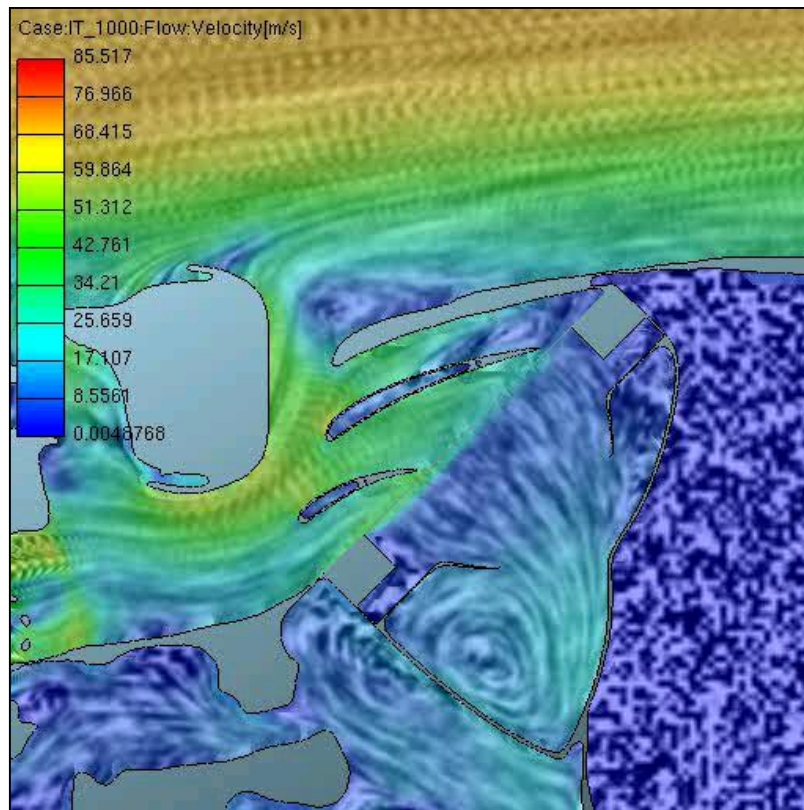


• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

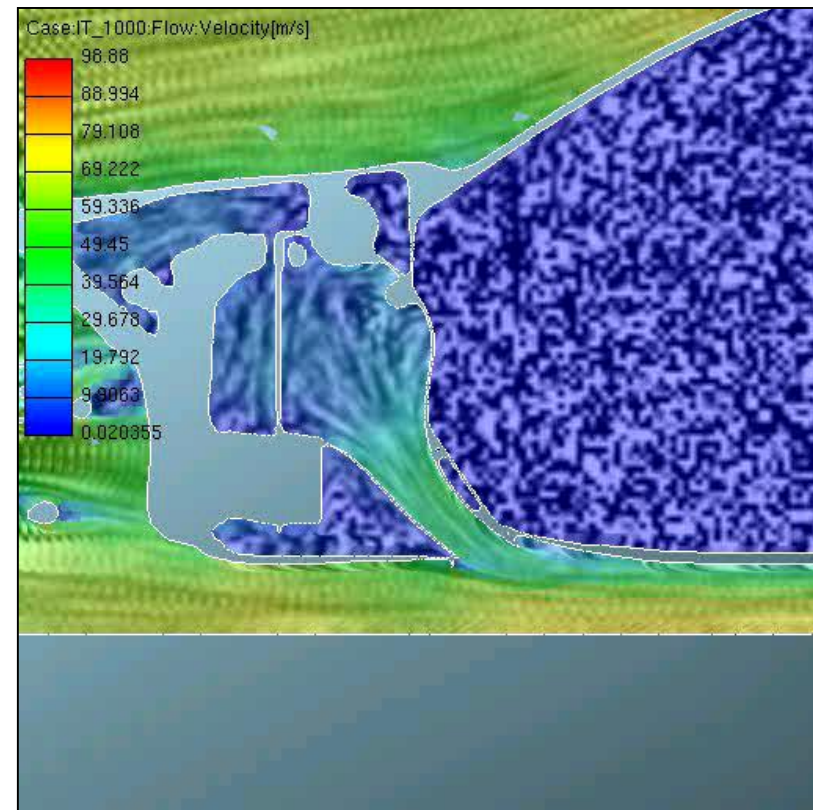
# CFD Results Optimization 4<sup>th</sup> Version (S)

## Videos

Flow around the vanes



Low pressure area underbody / venturi effect

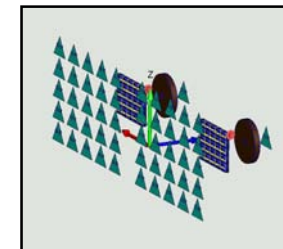
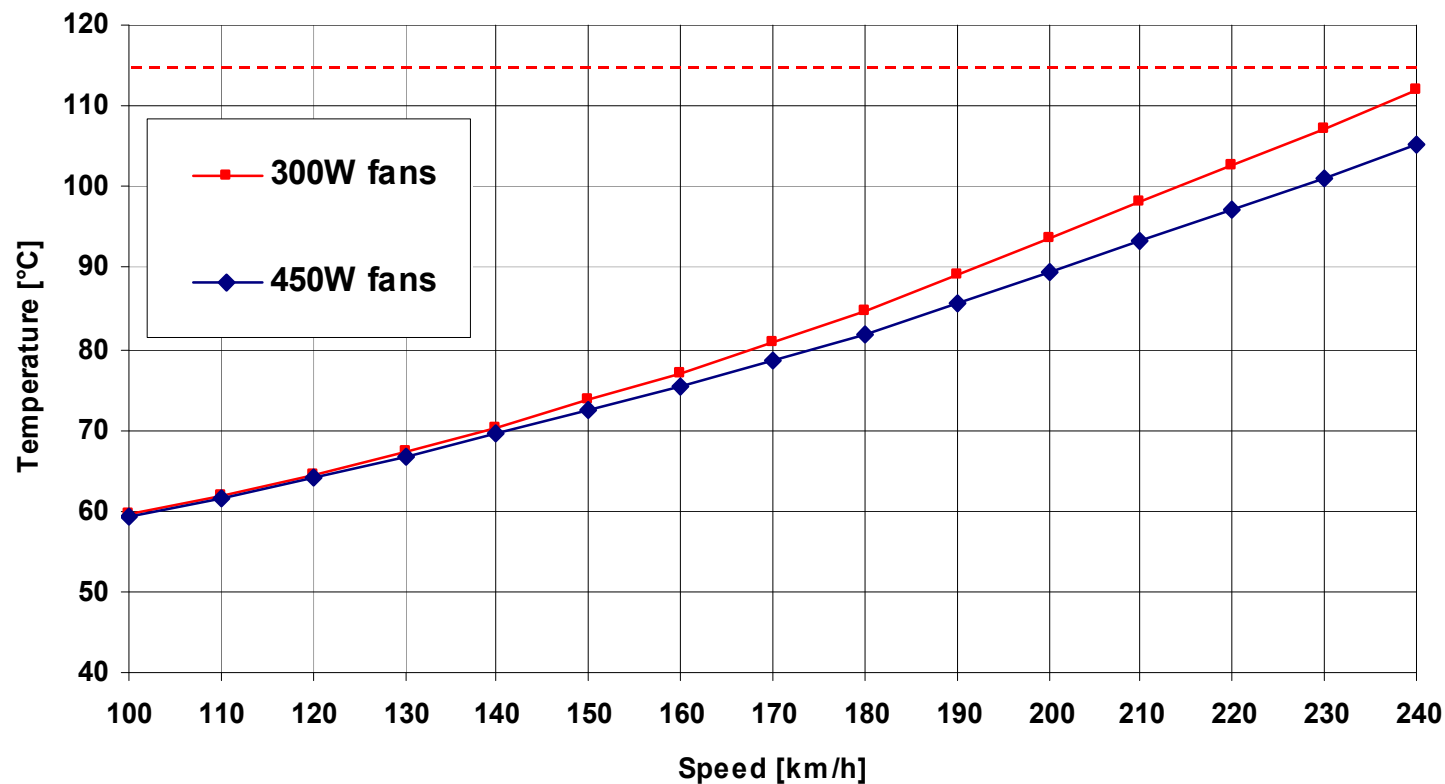


• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

# KULI Results Optimization 4<sup>th</sup> Version (S)

## Cooling Performance (Top Tank Temperature)

1D simulation results calculated with 450 W and 300 W fans, 45°C ambient temperature and a radiator with 32 mm depth.



• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

# KULI Results Optimization 4<sup>th</sup> Version (S)

## Cooling Performance, Summary 1D-Calculation

| KULI Simulation |   | 450W Electrical Fan       |  |  |   |  |   |   |   |
|-----------------|---|---------------------------|--|--|---|--|---|---|---|
|                 |   | responsible for modelling | Idle 1000 RPM<br>outside: 45°C<br>Limit: 115°C | Hill 7% with<br>60 km/h<br>outside: 35°C<br>Limit: 105°C | Hill 7% with<br>60 km/h and<br>trailer<br>outside: 35°C<br>Limit: 115°C | Hill 6% with<br>60 km/h<br>outside: 35°C<br>Limit: 105°C | Hill 6% with<br>90 km/h and<br>trailer<br>outside: 35°C<br>Limit: 115°C | Vmax 240<br>km/h<br>outside: 45°C<br>Limit: 113°C | Vmax with<br>trailer 180<br>km/h<br>outside: 45°C<br>Limit: 115°C |
| A               | Base version without any measures   | Christian Kussmann        | max. heat<br>54 kW                             | 57.1   | 67.4  | 65.3   | 78.9  | 181.2   | 118.1   |
| C               | New underhood with diffusor, condenser in front and vertical inlet fins                   | Christian Kussmann        | max. heat<br>64 kW                             | 50.7   | 59.6  | 58.3   | 70.1  | 159.9   | 106.3   |
| R               | New underhood with diffusor, condenser in front and vertical inlet fins and duct          | Christian Kussmann        | max. heat<br>64 kW                             | 54.2   | 65.5  | 60   | 72.9  | 107.1   | 89.5  |
| S               | New underhood with diffusor, condenser in front and modified vertical inlet fins and duct | Christian Kussmann        | max. heat<br>64 kW                             | 54.1   | 65.2  | 59.6   | 72.2  | 105.4   | 88.4  |
| KULI Simulation |   | 300W Electrical Fan       |  |  |   |  |   |   |   |
|                 |   | responsible for modelling | Idle 1000 RPM<br>outside: 45°C<br>Limit: 115°C | Hill 7% with<br>60 km/h<br>outside: 35°C<br>Limit: 105°C | Hill 7% with<br>60 km/h and<br>trailer<br>outside: 35°C<br>Limit: 115°C | Hill 6% with<br>60 km/h<br>outside: 35°C<br>Limit: 105°C | Hill 6% with<br>90 km/h and<br>trailer<br>outside: 35°C<br>Limit: 115°C | Vmax 240<br>km/h<br>outside: 45°C<br>Limit: 113°C | Vmax with<br>trailer 180<br>km/h<br>outside: 45°C<br>Limit: 115°C |
| A               | Base version without any measures   | Christian Kussmann        | max. heat<br>50 kW                             | 59.3   | 70.7  | 68   | 82.8  | 186.2   | 123.7   |
| C               | New underhood with diffusor, condenser in front and vertical inlet fins                   | Christian Kussmann        | max. heat<br>61 kW                             | 51.8   | 61.4  | 59.8   | 72.7  | 164.6   | 110.4   |
| R               | New underhood with diffusor, condenser in front and vertical inlet fins and duct          | Christian Kussmann        | max. heat<br>61 kW                             | 51.5   | 61.1  | 57.8   | 69.4  | 113.9   | 91.5  |
| S               | New underhood with diffusor, condenser in front and modified vertical inlet fins and duct | Christian Kussmann        | max. heat<br>61 kW                             | 51.4   | 61  | 57.5   | 69  | 111.9   | 90.6  |

• Introduction • Workflow/Validation • Scope of Work • Model Setup • Results • **Optimization** • Conclusions

## Conclusions

### Why coupling KULI to CFD?

- For new cooling concepts it is essential to couple KULI to CFD to get feasible results.
- Without the coupling it would not have been possible to show what is necessary to fulfill the requirements and to optimize the system in the very early stage.
- With the  $c_p$  matrix coupling, it was possible to represent 3D flow effects in the KULI model and even back flow is shown.
- Finally, version 4 seems to be a feasible solution for this vehicle.
- The final results show, that with a better air guidance even with 300 W fans a sufficient cooling of the system is achieved.

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