

Mobile HVAC-System with CO₂ as Refrigerant

Results of Simulation and Measurements

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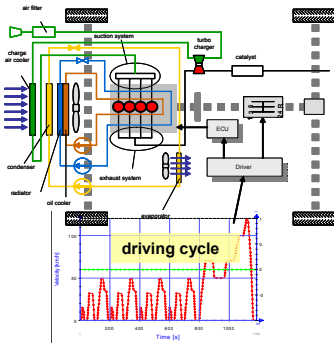
KULI User Meeting

29. 6. - 1. 7. 2005, Steyr, Austria

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(Steady state, driving cycle)
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Introduction



- HVAC-system is part of thermal network
- Influenced by / influences
 - underhood flow
 - engine operating point
 - fuel consumption, emissions
 - comfort
- Rarely running at steady-state operating conditions

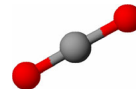
HVAC-System

- (transient) simulation of the HVAC-system
- Supplementary heating
- Effect on vehicle
- Refrigerant CO_2

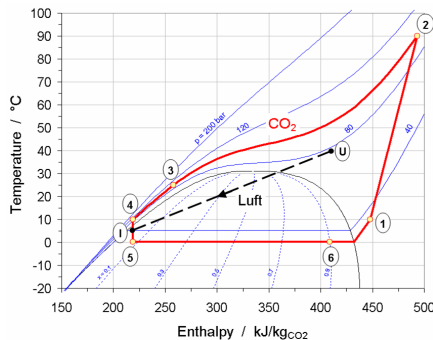
Introduction

Why CO_2 ?

- R134a-phase out due to high GWP (1300!)
- Most promising alternatives:
 CO_2 (Carbone Dioxid, R744), R152a



CO_2

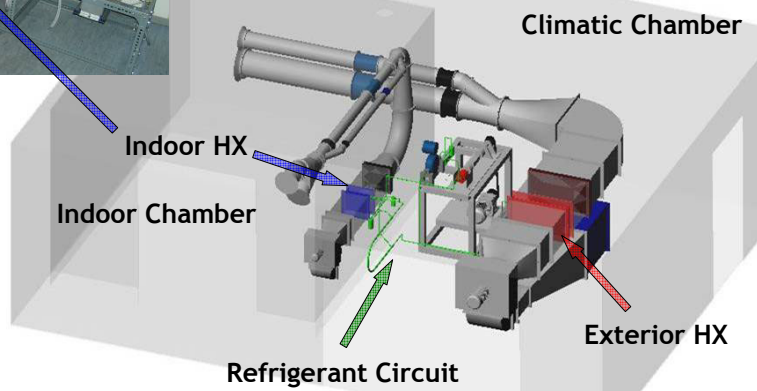


- Important differences to R134a
 - Critical point: 31.1°C / 73.8 bar
 - Supercritical refrigerant cycle (no condensation in „Condenser“)
 - High volumetric cooling capacity
- New components
- Different refrigerant circuit layout
- ➔ Measurements & Simulations for circuit optimisation

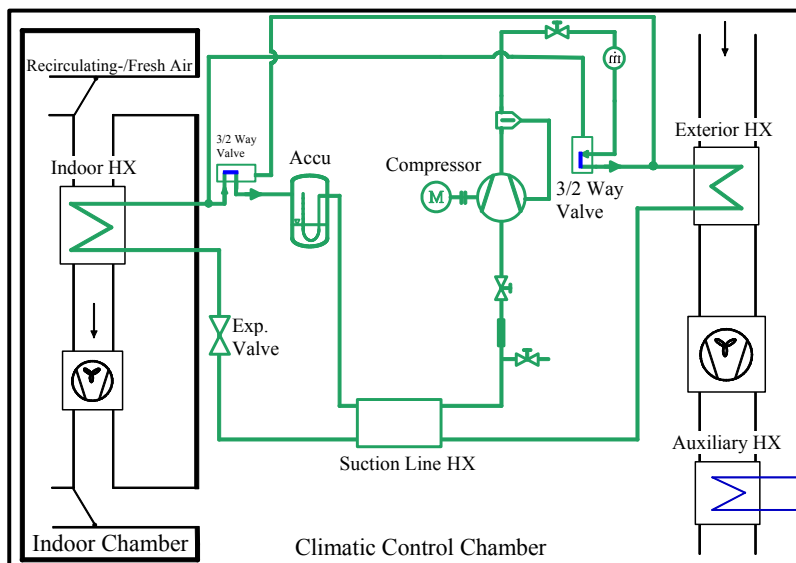
HVAC Test Rig



- Ambient Temperature: -20 .. +40 °C
- Ambient Humidity: 20 80 %
- Volume Flow Rate Indoor HX: 60 500 m³/h
- Volume Flow Rate Exterior HX: 600 .. 4000 m³/h

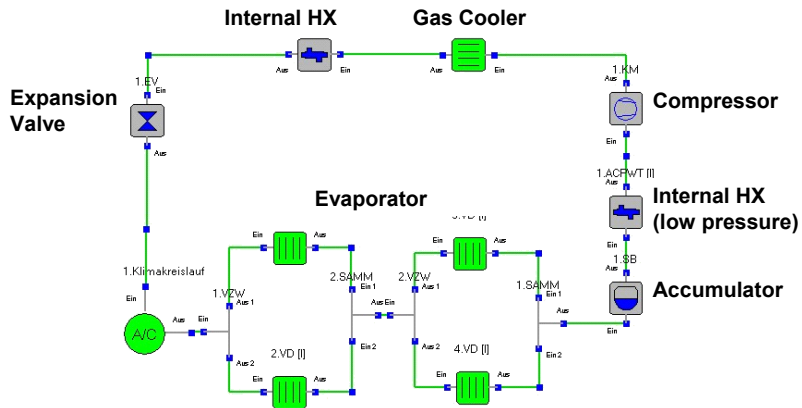


HVAC Test Rig



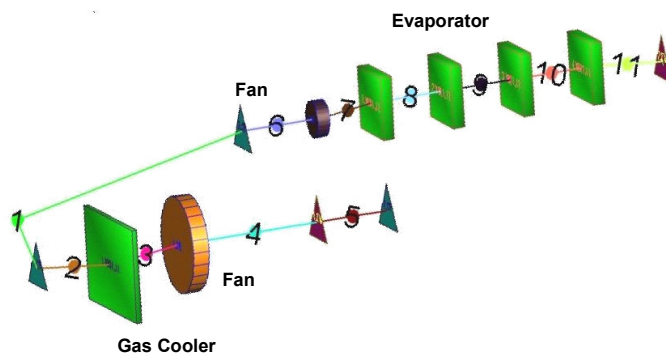
KULI-Simulations - Basics

Model of Refrigerant Circuit - AC



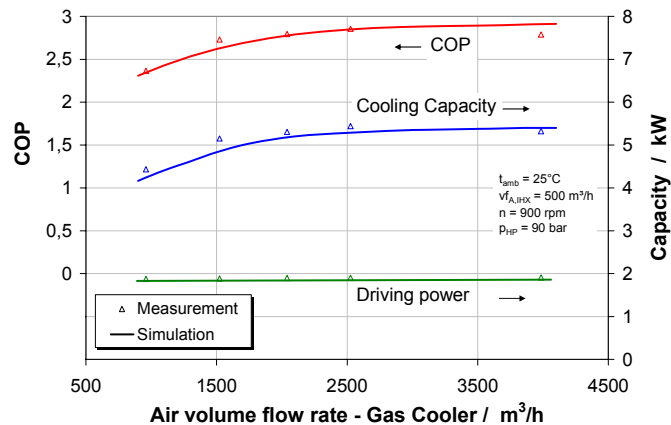
KULI-Simulations - Basics

Model of Air Side Flow



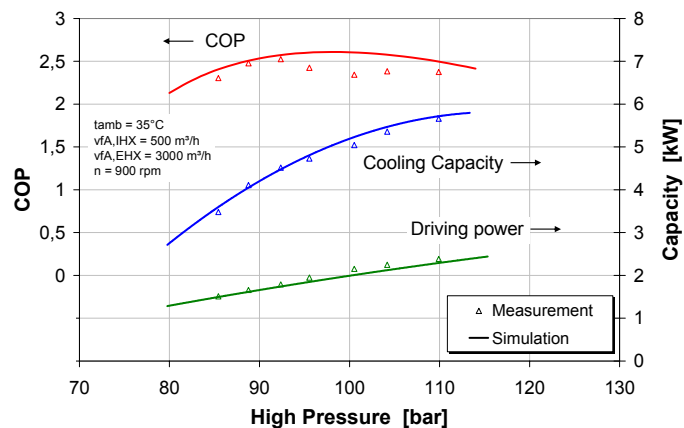
KULI Simulation – steady state

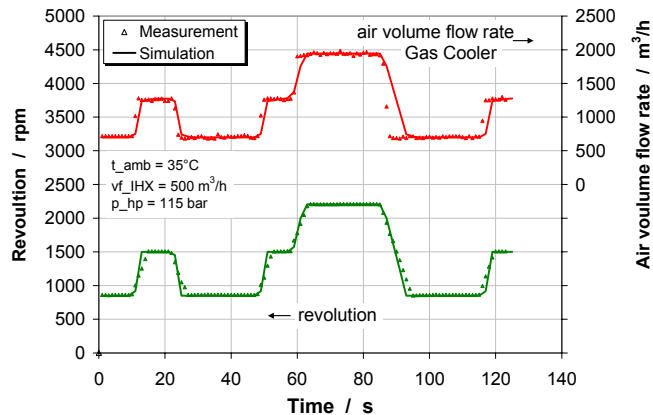
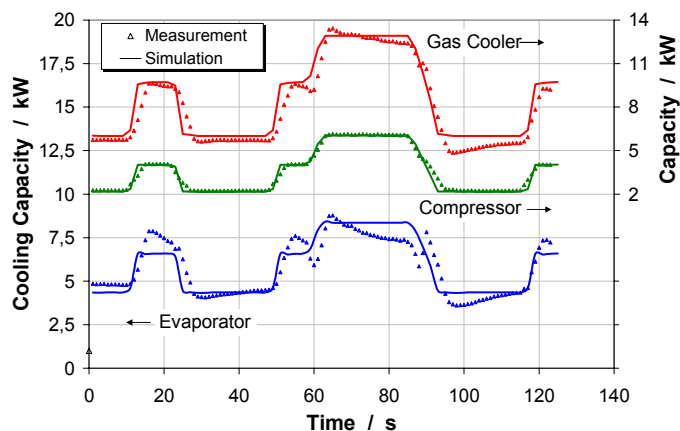
COP, Cooling Capacity and Driving Power vs. Gas Cooler Air Flow Rate



KULI Simulation - steady state

COP, Cooling Capacity and Driving Power vs. High Pressure



KULI Simulation – transient (quasi-steady state)**Driving cycle - Input variables****KULI Simulation – transient (quasi-steady state)****Driving cycle - Simulation vs. Measurement**

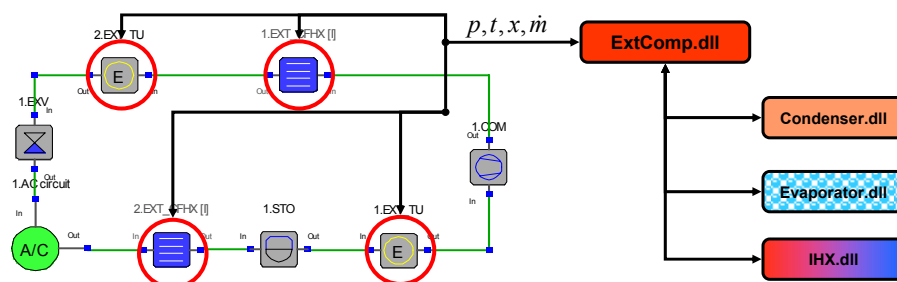
Advanced models

Requirements/Demands:

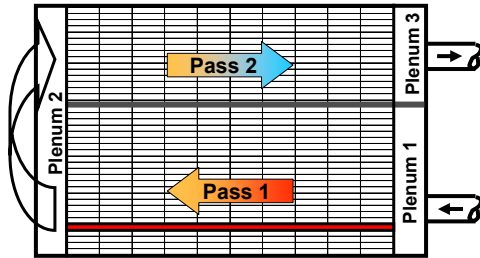
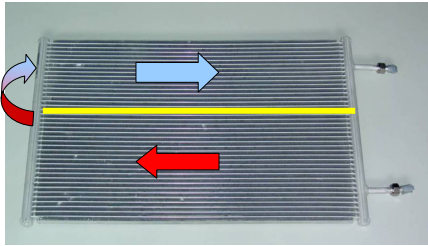
- Advanced geometry (modeling of various multi-pass geometry)
- Applicable for heating and cooling mode
- Transient behavior (structure warm-up)
- Airside coupling with CFD-simulation (non uniform airflow distribution)
- Implementation as External Component, coupling to KULI via enhanced COM-Interface

Coupling with KULI

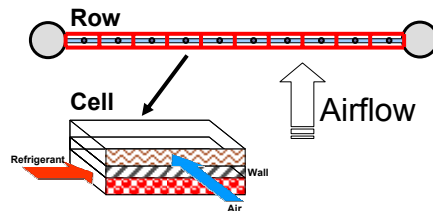
- Geometry Data from ASCII-File
- Component Results available through KULI-Postprocessor
- Detailed calculation results written to Log-File



Gas Cooler Model



- Transient formulation of mass- and energy conservation for each cell
- Heat transfer & pressure loss equations for single- and 2-phase flow (evaporation & condensation)
- Arbitrary cell length for high resolution



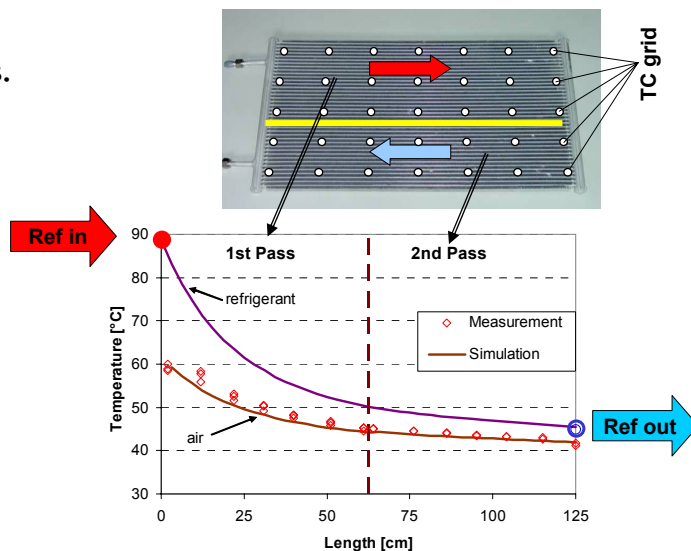
Gas Cooler – steady state

Measurement vs. Simulation

Inlet Conditions:
 $p_{\text{RefIn}} = 98 \text{ bar}$
 $T_{\text{RefIn}} = 89 \text{ }^{\circ}\text{C}$
 $\dot{m}_{\text{RefIn}} = 160 \text{ kg/h}$

$T_{\text{AirIn}} = 37.6 \text{ }^{\circ}\text{C}$
 $\dot{V}_{\text{Air}} = 2030 \text{ m}^3/\text{h}$

Rejected Heat:
 $Q = 5.9 \text{ kW}$



Gas Cooler – Warm Up

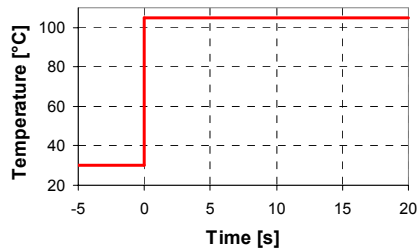
Gas Cooler warm up with / without
consideration of thermal mass

Refrigerant (CO₂) Inlet

$p = 110 \text{ bar}$

$\dot{m} = 200 \text{ kg/h}$

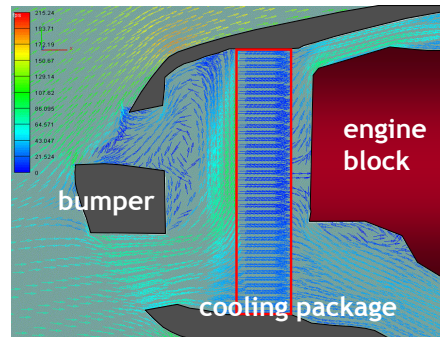
$t = 30^\circ\text{C} \rightarrow 105^\circ\text{C}$



Air (uneven distribution)

$V_f = 930 \text{ m}^3/\text{h}$

$t = 30^\circ\text{C}$

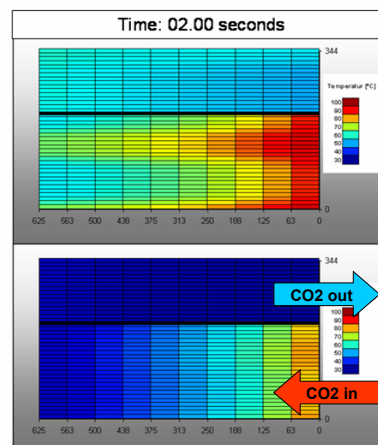
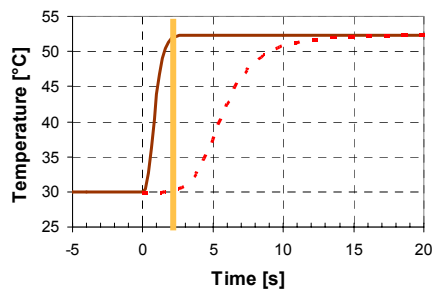


Gas Cooler – Warm Up

Surface temperature

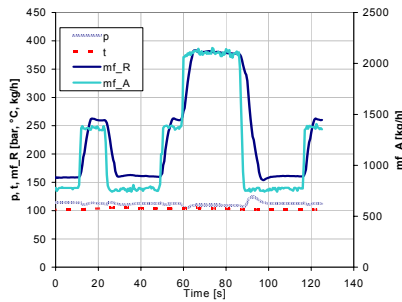
Refrigerant Outlet Temperature

— thermal capacity of gascooler neglected
- - - thermal capacity of gascooler considered



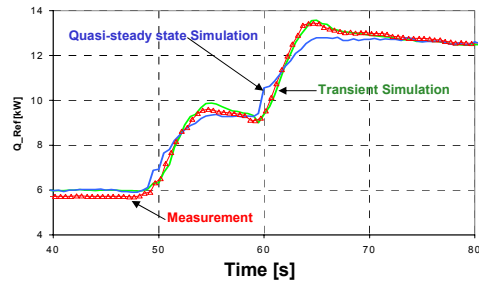
Advanced models – transient (Load Change)

Input variables



Gas Cooler heat rejection

$$\dot{H}_{\text{Ref,in}} - \dot{H}_{\text{Ref,out}}$$

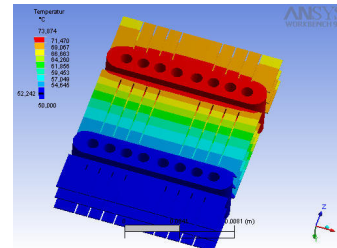


Summary

- Test rig for CO₂-HVAC system
- Influence of several system parameters investigated
- Good correlation of KULI simulation with measured data for steady state operation
- Advanced models
 - Simulation of component warm up
 - Gas cooler model: Good correlation with measurement data under transient conditions

Outlook

- Further improvement of HX-models
(heat conduction, refrigerant distribution, oil ...)
- Transient cycle analysis for heating & cooling mode (with Advanced Component Models)
- Simulations & experiments for heat pump mode
- Different set-ups of hp



Heat conduction in fins

Acknowledgement

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K plus Competence Center

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